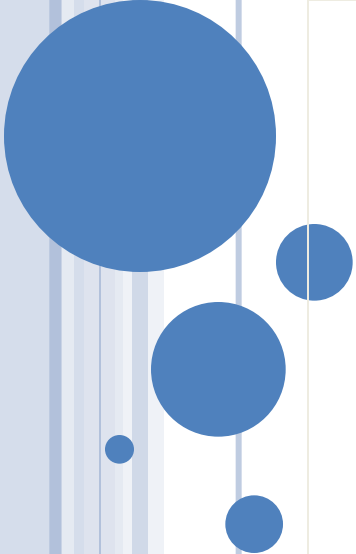




DESIGN IMPROVEMENTS OF THE W-BASED DIVERTOR CONCEPTS

X.R. Wang¹, S. Malang², M. S. Tillack¹, J. Burke

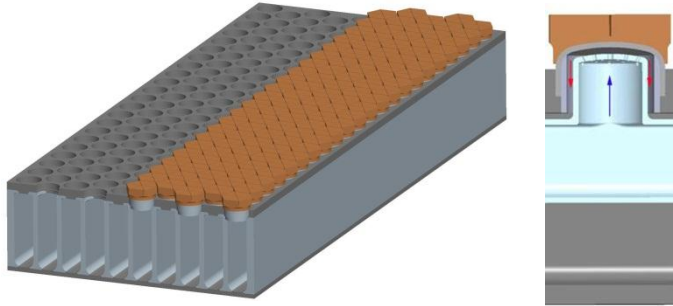
¹University of California, San Diego, CA

²Fusion Nuclear Technology Consulting, Germany

**ARIES-Pathways Project Meeting
Bethesda, MD
April 4-5, 2011**

STATUS OF THE HE-COOLED W DIVERTOR CONCEPTS

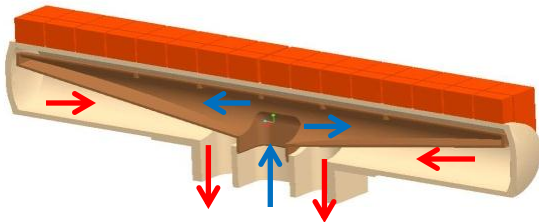
1. HCPF(He-cooled Combined Plate and Finger)



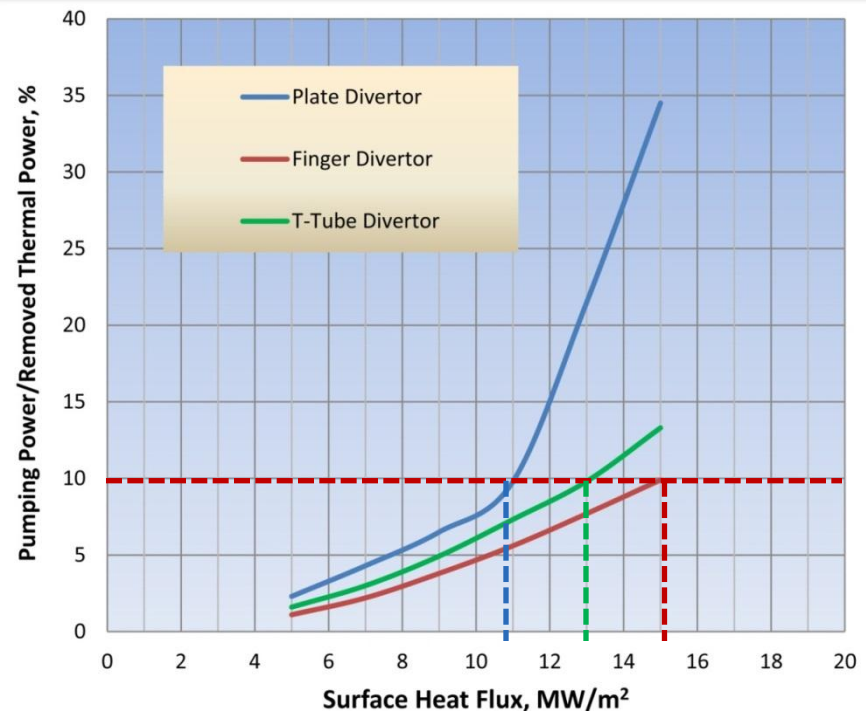
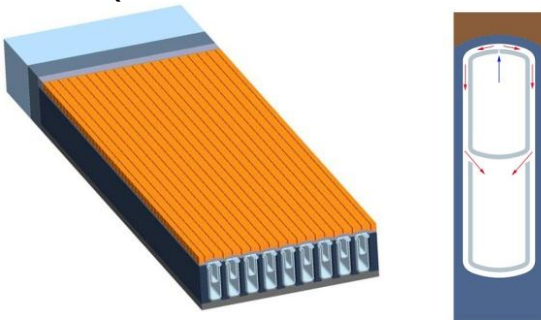
Design limits :

- ✓ Minimum temperature of the W structure 700 °C (embrittlement)
- ✓ Maximum temperature of the W structure 1300 °C (recrystallization)
- ✓ Pumping power 10% of thermal power removed.

2. HCTT(He-Cooled T-Tube)



3. HCFP(Helium-cooled Flat Plate) divertor

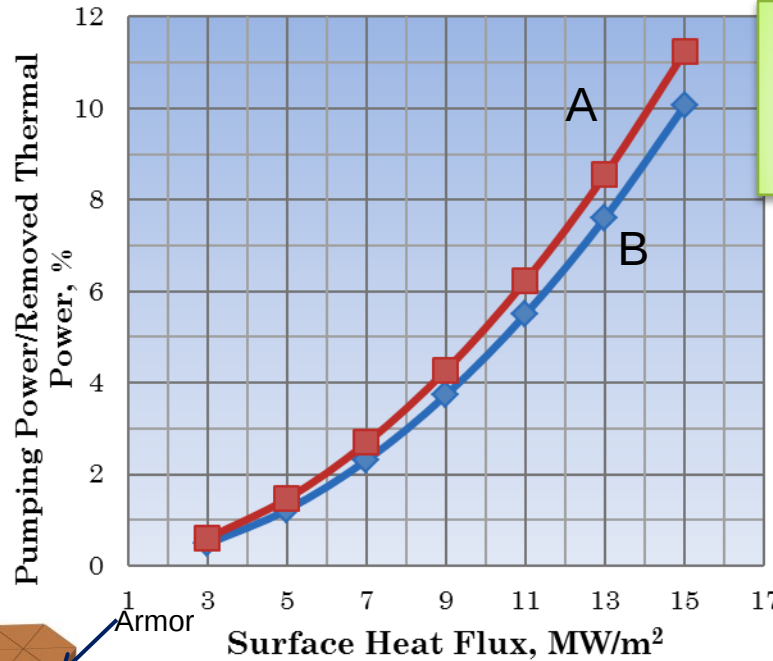


MODIFICATIONS

- Since fracture toughness of the W alloy is reduced by low temperature irradiation, the lower temperature limit ($\sim 700\text{ }^{\circ}\text{C}$) must probably be raised to $\sim 800\text{ }^{\circ}\text{C}$ to avoid the embrittlement of the W structure.
- To achieve this, the He coolant exit temperature has to be raised to a range of $750\sim 800\text{ }^{\circ}\text{C}$ (previous cooling conditions: $T_{\text{exit}}=650\sim 700\text{ }^{\circ}\text{C}$, $T_{\text{inlet}}=600\text{ }^{\circ}\text{C}$).
- Advanced ODS steel (such as 12CrYWTi) would be one option for the manifold. Creep tests of the 12CrYWTi at $800\text{ }^{\circ}\text{C}$ and 138 MPa showed the specimen is in the steady-state stage at $\sim 5000\text{ h}$ and ruptured at $\sim 14500\text{ h}$.*
- Such a higher He exit temperature requires a reduction in the maximum heat flux in order to keep the maximum temperature of the W structure under the upper operating limit of $1300\text{ }^{\circ}\text{C}$.
- We should make new optimizations to the divertor concepts with a goal to find the maximum allowable heat flux with obeying the $1300\text{ }^{\circ}\text{C}$ limit for the W structure and 10% limits for pumping power.
 - ✓ HCPF(Finger)
 - ✓ HCFP(Plate)
 - ✓ HCTT(T-Tube)

*R.L. Klueh et al./J. of Nuclear Materials 307-311 (2002) 455-465.

REEVALUATION OF THE HCPF CONCEPT



$P_{\text{pumping power}}/P_{\text{removed power}} < 10\%$
 A: $800 < T_s < 1300$ °C, $T_{\text{exit}} = 800$ °C, $T_{\text{in}} = 700$ °C
 B: $700 < T_s < 1300$ °C, $T_{\text{exit}} = 700$ °C, $T_{\text{in}} = 600$ °C

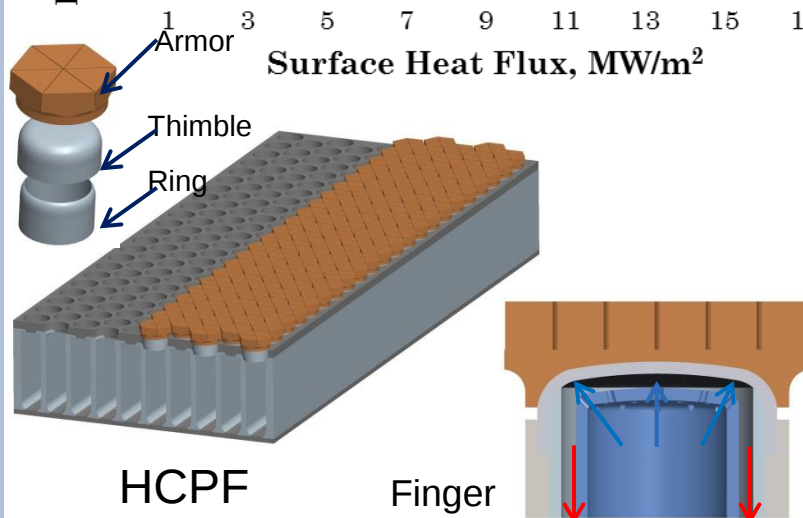
- The allowable heat flux calculated by CFX
- ✓ **Case A, ~14 MW/m²**
- ✓ **Case B, ~15 MW/m²**

Material applications:

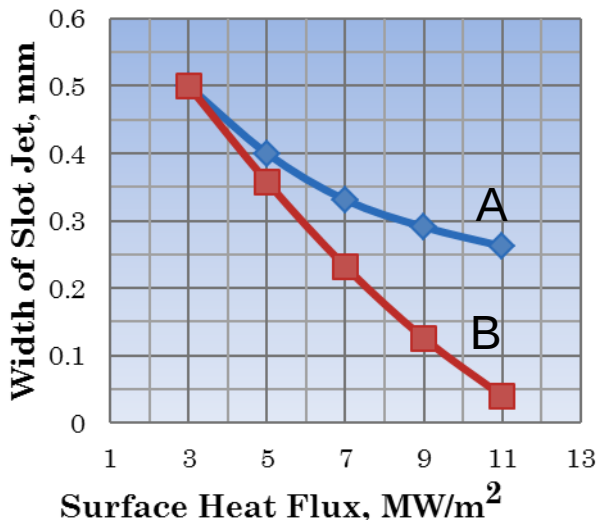
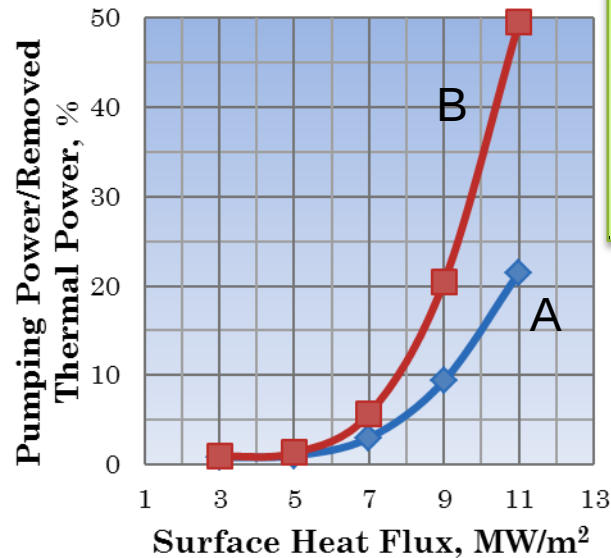
- ✓ Armor: pure W
- ✓ Cup-shaped thimbles: VM-W doped with ppm of $\text{Al}_2(\text{SiO}_3)_3$ or K_2SiO_3
- ✓ Front, side and back plates: WL10 ($\text{W-1\%La}_2\text{O}_3$)

Fabrication:

- ✓ Armor: powder HIPing(hot isostatic pressing)
- ✓ Thimbles: Drawn and rolled from sheet (up to 1 mm)
- ✓ All the plates: HIP



REEVALUATION OF THE HCFP DIVERTOR



$P_{\text{pumping power}} / P_{\text{removed power}} < 10\%$

A: $800 < T_s < 1300$ °C, $T_{\text{exit}} = 800$ °C, $T_{\text{in}} = 700$ °C

B: $800 < T_s < 1200$ °C, $T_{\text{exit}} = 800$ °C, $T_{\text{in}} = 700$ °C

C: $700 < T_s < 1300$ °C, $T_{\text{exit}} = 677$ °C, $T_{\text{in}} = 600$ °C

➤ The allowable heat flux calculated by CFX

✓ Case A, ~9 MW/m²

✓ Case B, ~7.8 MW/m²

✓ Case C, ~11 MW/m²

Material applications:

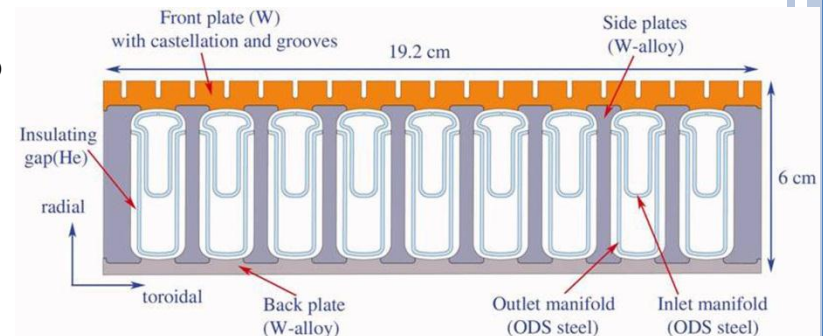
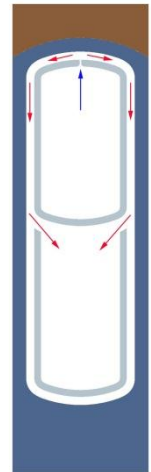
✓ Armor: pure W

✓ Front, side and back plates
(thickness > 2mm)

Pure W or WL10 (W-1%La₂O₃)

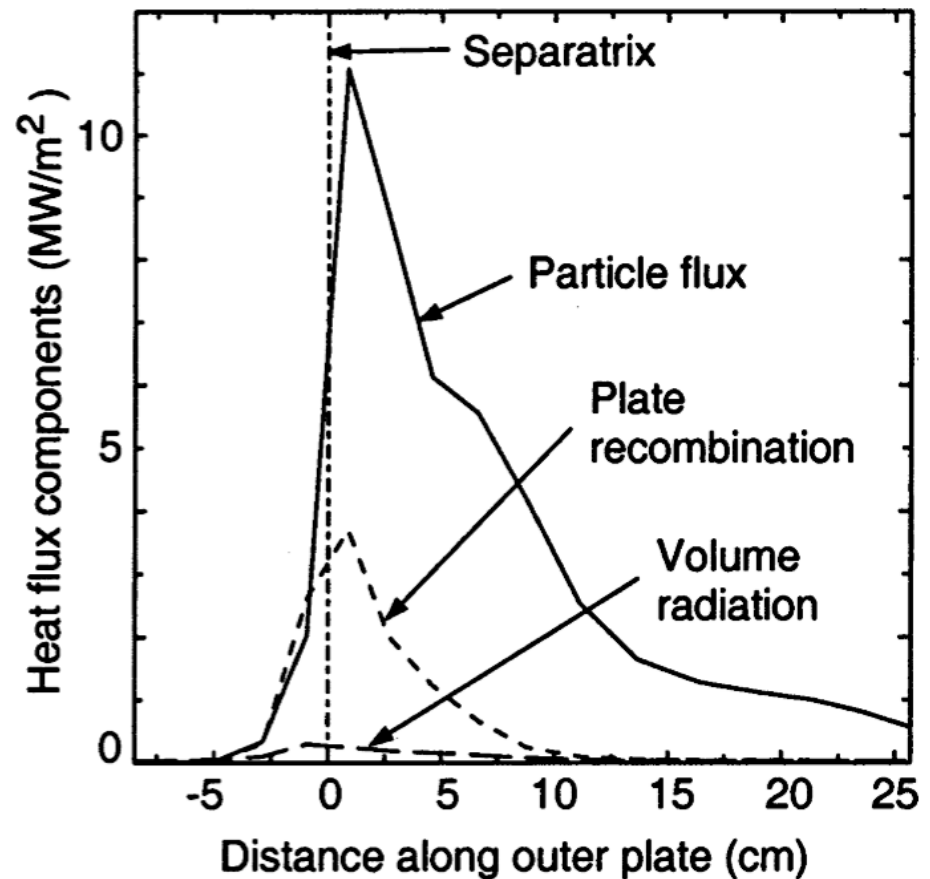
Fabrication:

✓ All plates: HIP



HOW CAN A NON-UNIFORM HEAT FLUX PROFILE BE HANDLED IN DIVERTOR DESIGN

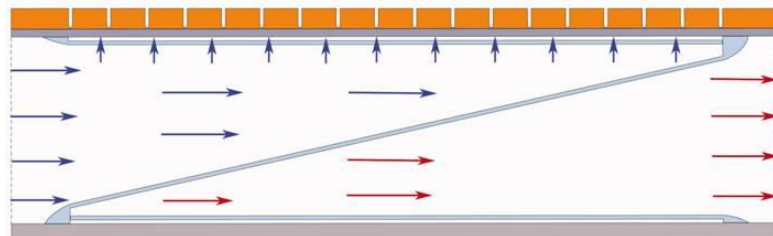
- In previous studies, all evaluations until now were concerned with the layout of the finger and plate concepts for the maximum allowable heat flux.
- However, there is a rather steep profile in the plate direction with a ratio between the maximum (11 MW/m^2) and average heat flux (3.5 MW/m^2) of 2~3.
- How can the localized HHF be accommodated for in our divertor design?



Example of the ARIES-AT or RS Design
*T.D. Rognlien, presented at the ARIES Town Meeting

ADJUSTMENT OF THE COOLING TO A POWER PROFILE IN PLATE DIRECTION

- With the finger, plate and T-Tube design, the coolant flows in poloidal direction in parallel channel. The heat flux, cooling condition, and the temperatures are equal in these parallel channels.
- The helium enters at one side of the plate and leaves at the other side, and the heat flux into the plate is low at the channel inlet, rises to a maximum value, and goes down again in the direction to the exit.
- Inside the channel there is an inlet manifold with decreasing cross-section, and an outlet manifold with increasing cross-section. With this arrangement, the He velocity in the inlet and outlet manifold remains constant over entire length of the plate.
- The **design method** for adjusting the cooling to a power profile in the plate direction is to maintain the He velocity through the nozzles (jets) as well as the coolant temperature rise ($T_{\text{exit}} - T_{\text{inlet}}$) as constant over the entire length of the plate. In this way, the pressure drop (ΔP) will remain constant.



DESIGN METHOD OF ACCOMMODATING TO A POWER PROFILE IN PLATE DIRECTION

To maintain the jet velocity v_{jet} , coolant temperate rise ΔT and the ΔP as the constant:

- We have to adjust the local coolant mass flow rate to the local divertor heat flux, and this requires **the ratio of the mass flow rate to surface heat flux should be constant**. This means we have to adjust the **flow cross-sections** to the local divertor heat flux.
- The ratio of the local cross-section area of the nozzle to the cross-section area of the nozzles at the location of the maximum heat flux has to the same ratio of the local heat flux to the maximum heat flux:

$$S_{\text{nozzle at local}} / S_{\text{nozzle at max}} = q''_{\text{local}} / q''_{\text{max}} \quad (S_{\text{nozzle}}: \text{CS area of the nozzles})$$

- The slot should not become too small for practical reason, the continuous slot should be replaced by an interrupted slot with ratio

$$L_{\text{slot at local}} / L_{\text{slot at max}} = q''_{\text{local}} / q''_{\text{max}} \quad (L_{\text{slot}}: \text{length of the slot})$$

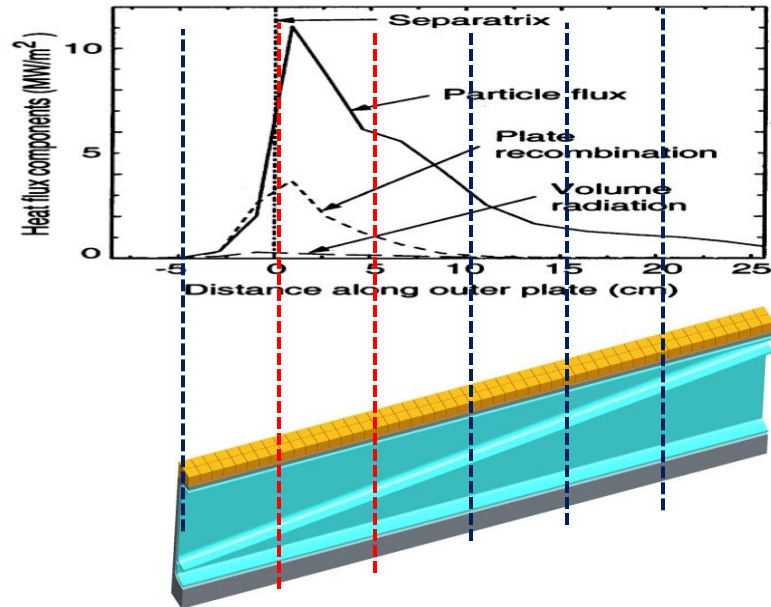
➤ Examples:

Plate design with jet cooling through a slot

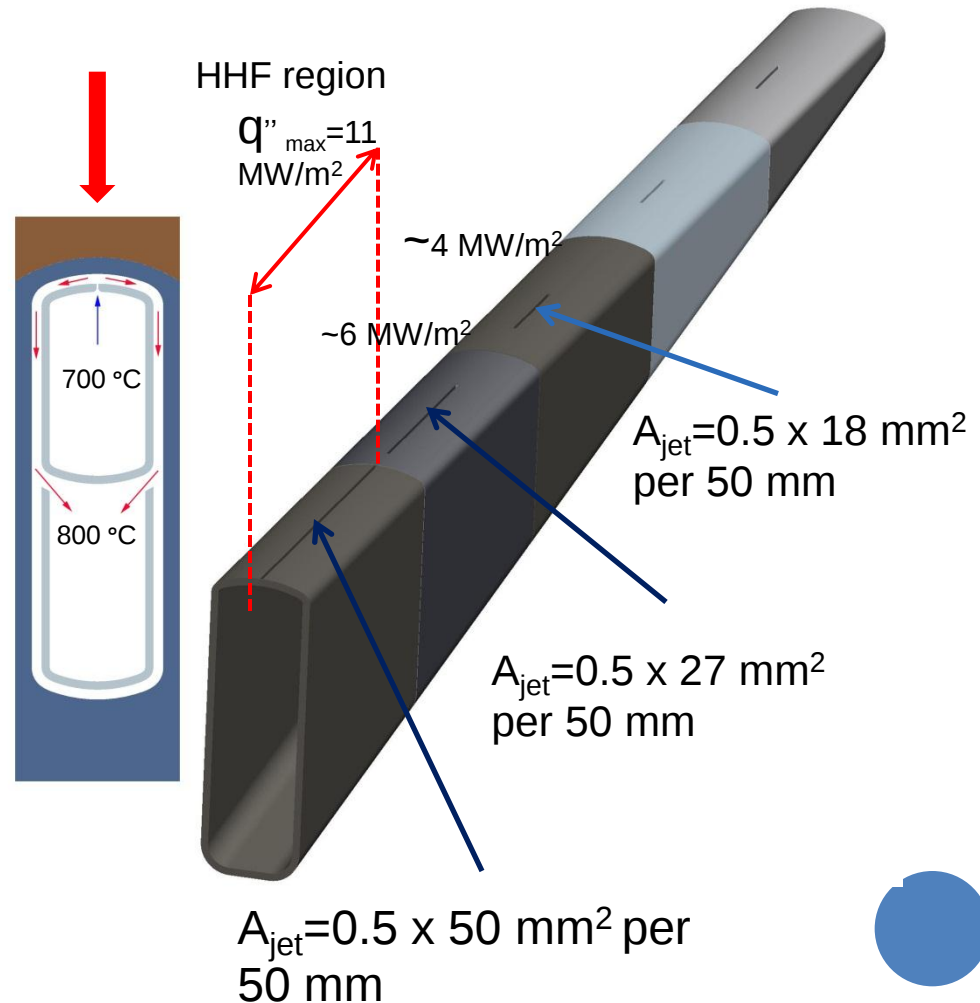
Heat flux (MW/m ²)	10	5	2
Slot length (mm per 5 mm unit)	5	2.5	1



EXAMPLE OF THE HCFP DIVERTOR TO ACCOMMODATE A HF LIKE ARIES-AT

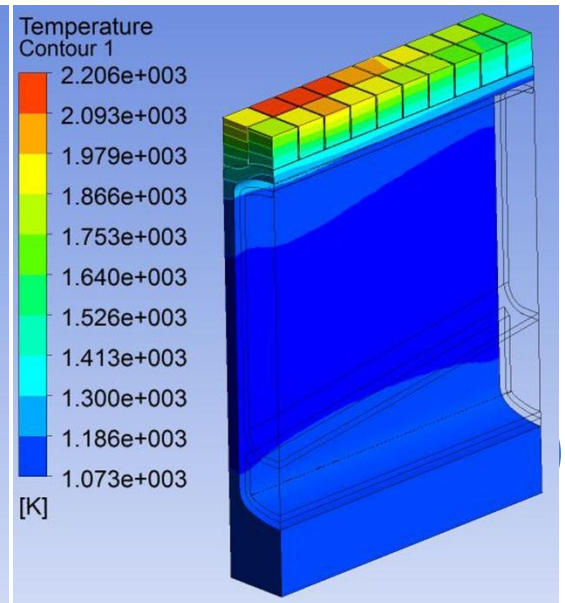
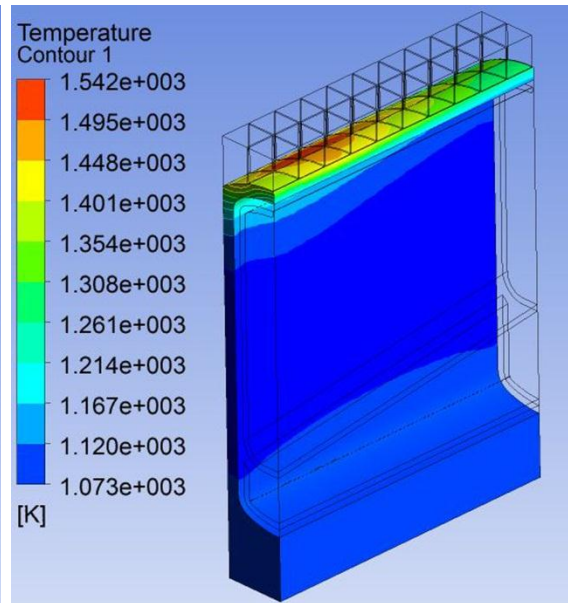
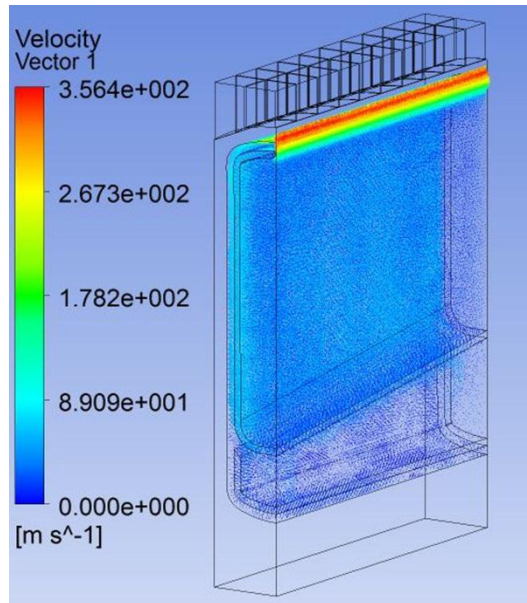


- HCFP: poloidal length ~30 cm
- High heat flux region ~5 cm in the poloidal direction
- V_{jet} , ΔT and ΔP remain constant

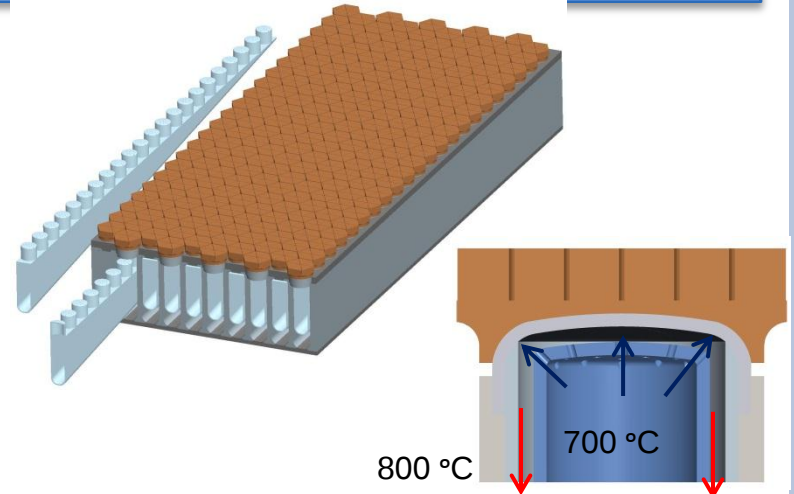
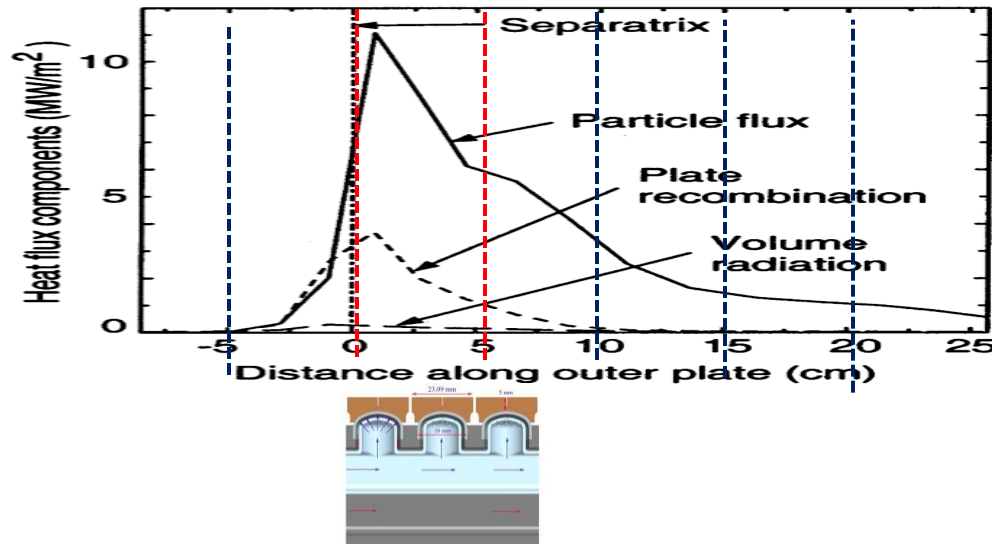


CFD RESULTS INDICATED THE PLATE CAN HANDLE THE ARIES-AT LIKE POWER PROFILE

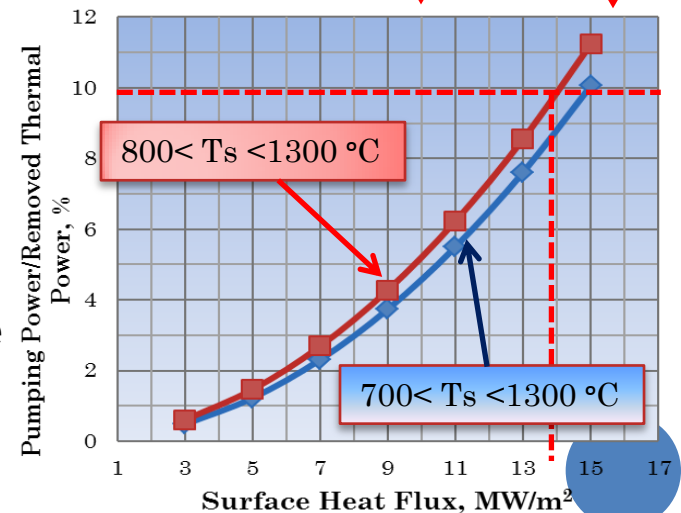
- Only high heat flux region(L=50 mm) with the maximum heat flux of 11 MW/m² is simulated by CFD code in order to reduce the elements and nodes.
- Total number of the nodes is ~4.1 million for the high heat flux region.
- CFD results are based on $T_{\text{exit}}/T_{\text{inlet}}=800/700$ °C and W structure operating temperature of $800 < T_s < 1300$ °C
 $V_{\text{jet}}=356$ m/s, $T_{s,\text{max}}=1296$ °C, $T_{s,\text{min}}=800$ °C, $T_{\text{armor,max}}=1933$ °C
 $P_{\text{pumping}}/P_{\text{removed thermal}}=9.6\%$



EXAMPLE OF THE HCPF TO ACCOMMODATE A POWER PROFILE LIKE ARIES-AT



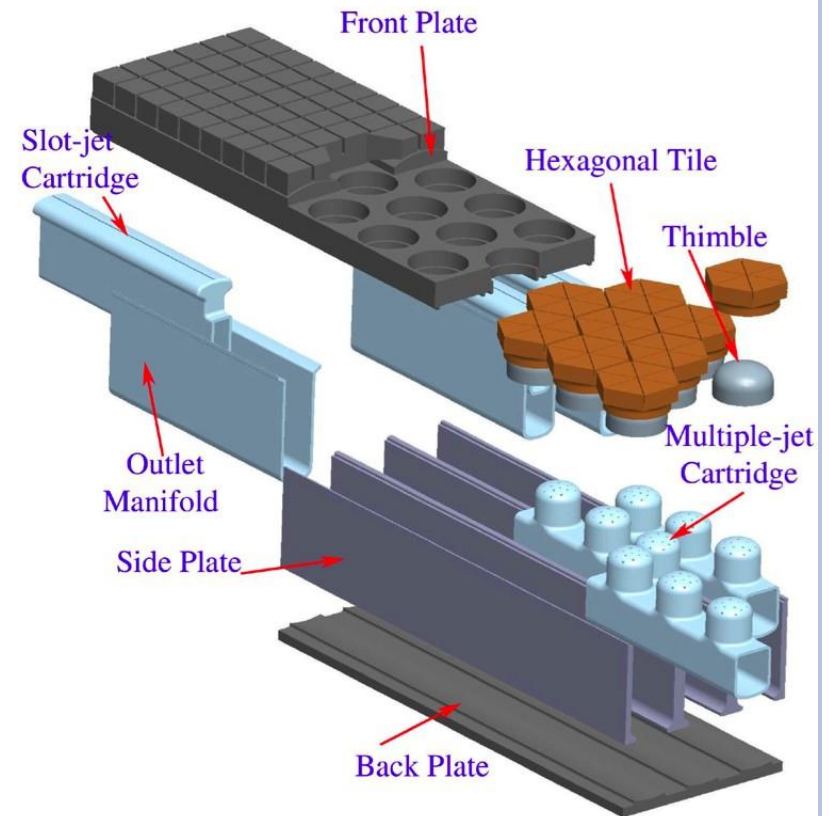
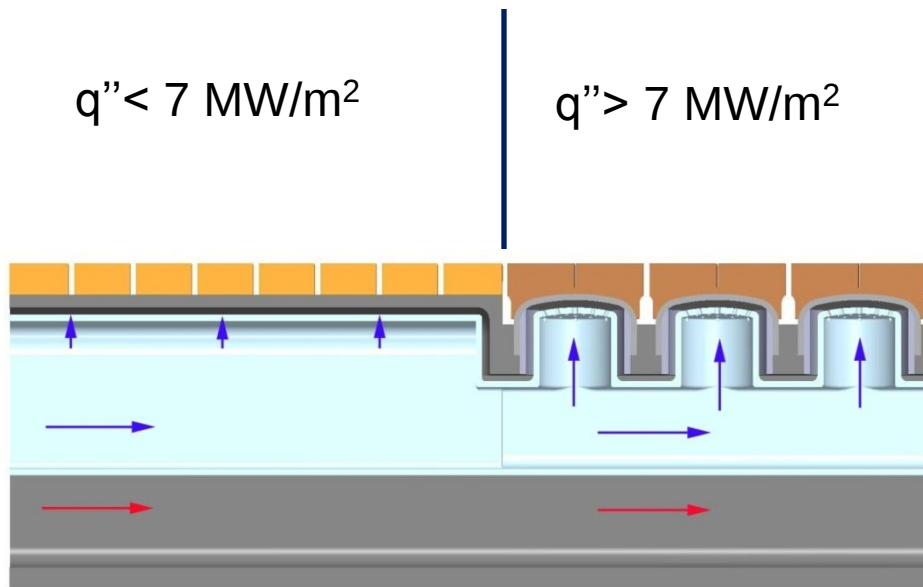
- HCPF: poloidal length ~30 cm
- ~2 to 3 fingers in the high heat flux region,
- Maintaining the V_{jet} , ΔT and ΔP as constant over whole poloidal length
- Keeping the nozzle diameter constant and decrease the number of nozzles in the same ratio as the local heat flux
- Accommodating much higher localized heat flux (about 6% only for $q''=11\text{MW/m}^2$)
- No analysis been done.



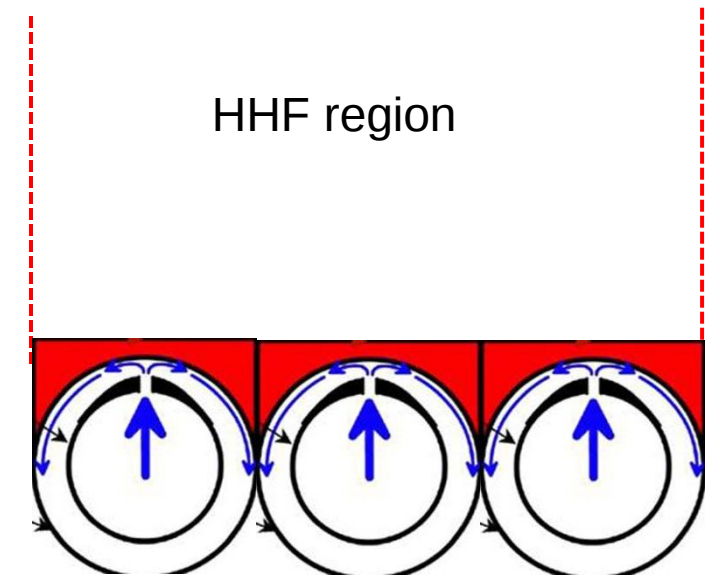
Uniform heat flux

FINGERS CAN BE COMBINED WITH SLOT JETS FOR LOCALIZED HHF HANDLING CAPABILITY AND MINIMUM UNITS

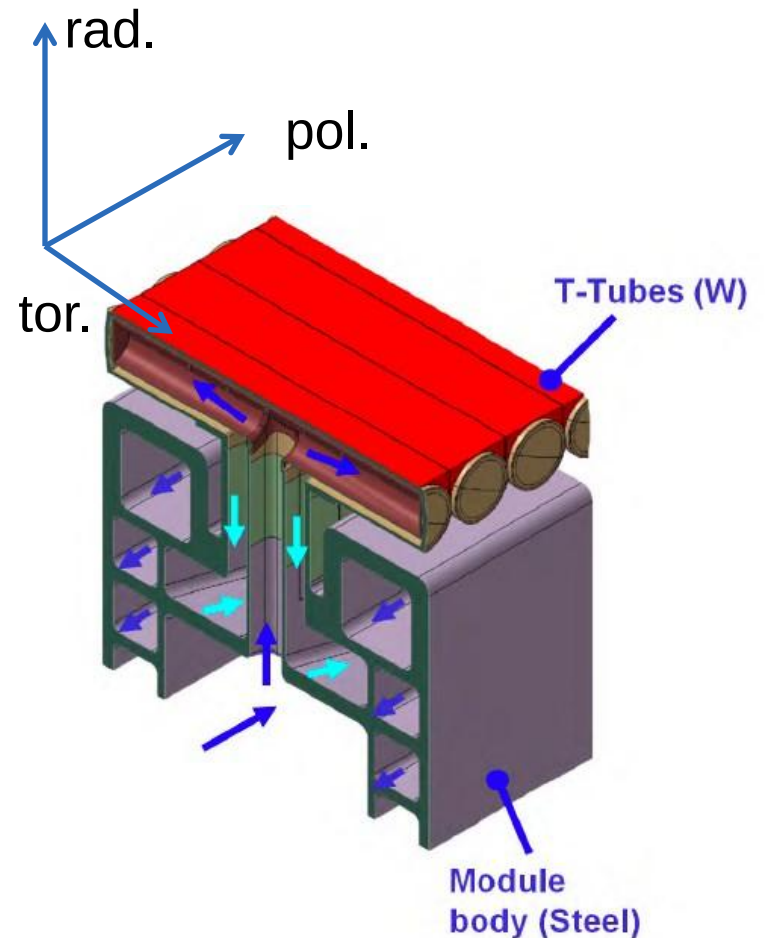
- Cartridges with both slots and fingers can be combined
- Limit fingers to zones with the highest heat flux



EXAMPLE OF THE HCTT TO ACCOMMODATE A POWER PROFILE LIKE ARIES-AT (JEREMY)



➤ V_{jet} , ΔT and ΔP remain constant



SUMMARY AND CONCLUSIONS

- Divertor concepts including the finger, T-Tube and plate-type, have been re-evaluated and optimized based on raising the minimum W temperature from 700 °C to 800 °C in order to avoid embrittlement under neutron irradiation.
- Allowable heat flux based on the operating temperature limits of $800\text{ °C} < T_s < 1300\text{ °C}$
 - ✓ HCPF(Finger), $q''_{\text{allow}} = 14\text{ MW/m}^2$
 - ✓ HCTT(T-tube), $q''_{\text{allow}} = ???$ (waiting for results from Jeremy)
 - ✓ HCFP(Plate), $q''_{\text{allow}} = 9\text{ MW/m}^2$
- Stresses for the adjustment of the W minimum temperature from 700 to 800 °C have not been checked, yet. However, thermal stress should be reduced because of the raising the W minimum temperature. ($\Delta T = 400\text{ °C}$ comparing to $\Delta T = 500\text{ °C}$).
- Previous studies indicate in all concepts, stress relaxation by plastic deformations reduce the stresses to a degree that the sum of primary and secondary stresses remain below the $3 S_m$ limit of the ASME code.
- Initial results indicate that the plate-type divertor can accommodate a localized HHF like the ARIES-AT with maximum heat flux of 11 MW/m^2 while meeting the design limits of the W-structure temperature and pumping power.
- **Expecting (without CFD analysis)** the finger can handle much higher localized HHF than the plate.