

Dual Coolant Blankets and Helium Cooled Divertor Targets

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Characteristics of the Dual Coolant Lead Lithium (DCLL) Breeding Blanket

- Helium cooled First Wall and blanket structure
- Self cooled lead lithium breeding zone
- SiC flow channel inserts serving as electrical and thermal insulator

Key issues determining the performance of a power plant equipped with DCLL blankets

- Degree of thermal insulation between flowing LM and steel structure
 - Maximum allowable interface temp. steel/lead lithium
 - Maximum allowable temperature of FW/blanket structure
 - Fraction of the total thermal power removed with the LM
- ➔ Achievable lead-lithium exit temperature
- Helium cooling conditions (pressure, flow rate, inlet/outlet temperature) required to remove FW surface heat flux and to meet the temperature criteria above
 - Required pumping power for the helium coolant
- ➔ Achievable thermal efficiency in the power conversion system

Crucial areas for the DCLL blanket development

- A) Flow channel inserts
- B) MHD heat transfer including impact of natural convection
- C) Fabrication of FW/blanket structure including ODS-plating at the FW
- D) Blanket integration including manifolding, connection to coolant access pipes, mechanical attachment

Requirements on Flow channel inserts

a) Sufficiently low electrical conductivity

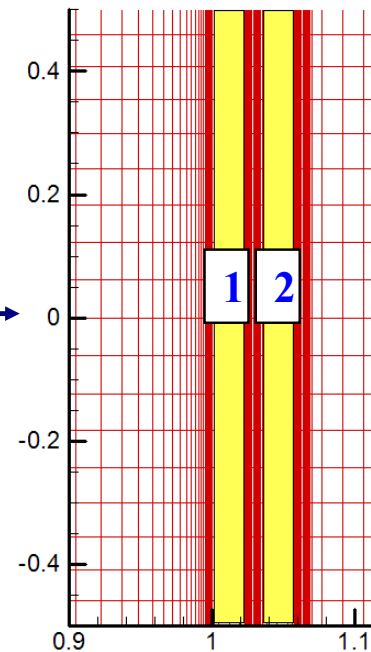
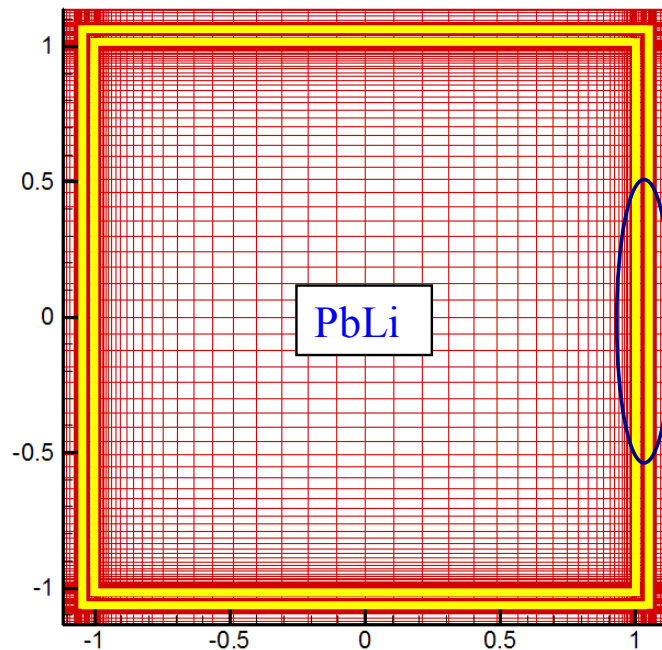
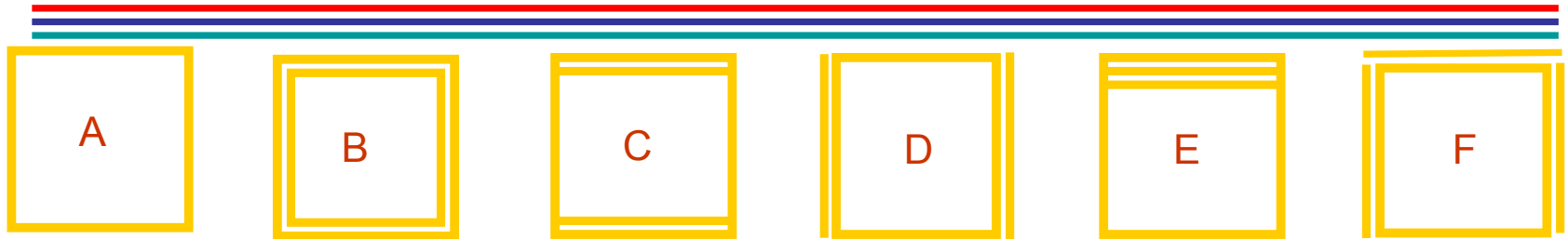
$$\sigma < 20 \text{ S/m}$$

b) Low thermal conductivity $k < 2 \text{ W/(m} \cdot \text{K)}$

c) Surfaces sealed to avoid “soaking” of LM

d) Sufficient strength to survive thermal stresses

Analysis for “nested” FCI



NESTED (2-layer) FCI

DEMO (current design)

Front duct: $2a=207$ mm;
 $2b=207$ mm
 FCI 1: 2.5 mm
 Gap 1: 1 mm
 FCI 2: 2.5 mm
 Gap 2: 1 mm

$\sigma_{\text{SiC}}=20$ S/m
 $k_{\text{SiC}}=1$ W/m-K

Various “nested” FCI options can be implemented. Here, Option B is considered.

E)

MHD heat transfer including impact of buoyant flow

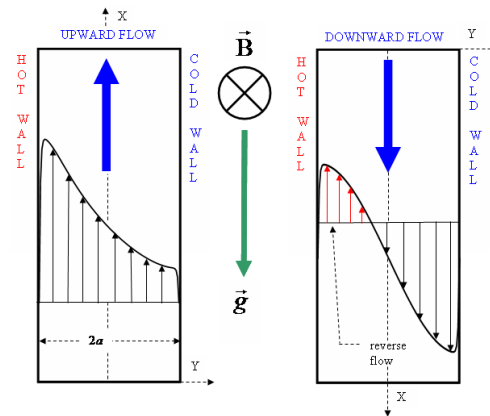
Analysis of buoyant flows

Motivation:

- In both ITER and DEMO, buoyant flows can be stronger than the forced flow. Buoyant flows will affect: heat losses, FCI ΔT , and the interface temperature.
- In the case where the forced flow is downwards, a “reverse” flow can appear at the “hot” wall. This may result in excessive local temperature increase in the PbLi.

Goals:

- To characterize velocity and temperature distributions in the poloidal ducts for two forced flow directions: upwards and downwards.
- To find conditions when the reverse flow appears (for downward flows).
- To apply the new theory to the poloidal flows in ITER and DEMO.



Parameter	ITER	DEMO IB	DEMO OB
Ha	6500	12,000	30,000
Re	30,000	60,000	40,000
Gr	7.0×10^9	3.5×10^{12}	2.0×10^{12}

Helium Cooled Divertor Targets

Why helium cooling and not water or liquid metal?

- **Cooling with (cold) water would not allow to use the divertor heat (10-15 % of total thermal power) in the power conversion system, copper heat sink as in ITER not possible for the large neutron fluence of a power plant,**
- **Liquid metal cooling would be limited to $\sim 5 \text{ MW/m}^2$ surface heat flux (plasma physics requires at least 10 MW/m^2)**

Crucial issues of helium cooled divertor targets

- *Exceptional high heat flux requires:*

- a) Structural material with high thermal conductivity (close to $1 \text{ W}/(\text{m} \cdot \text{K})$)
- b) Heat transfer coefficient helium-wall at least $30,000 \text{ W}/(\text{m}^2 \cdot \text{K})$

- *Structural material required with a sufficiently large temperature window:*

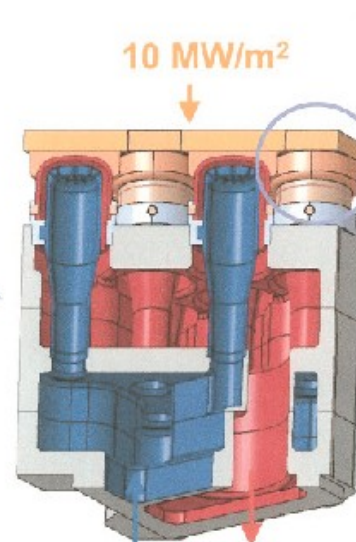
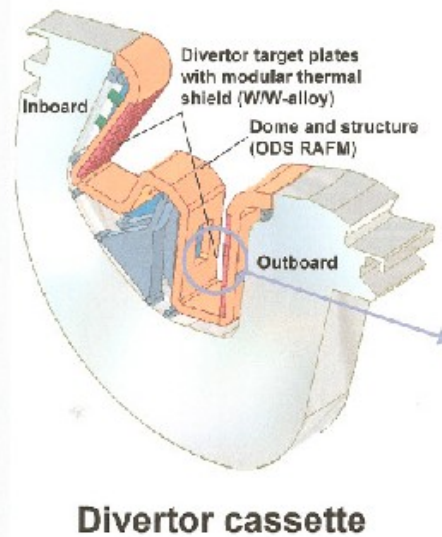
- a) Minimum temperature above embrittlement limit
- b) Maximum temperature below recrystallization limit

- *Thermal stresses and target deformation in an allowable range:*

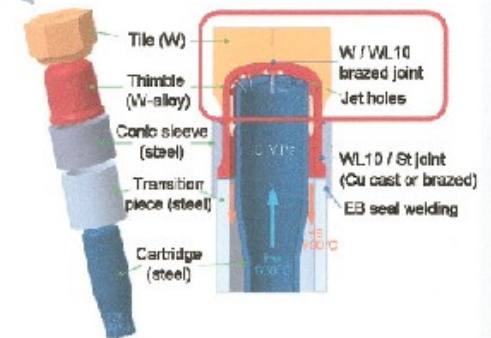
- a) High heat flux causes large temperature gradient in the target ($100 \text{ K}/\text{mm}$)
 - > large temperature differences across target plate
- b) Thermal stresses and deformations caused by these large temperature differences depend on the size of target and the kind of mechanical support : Alternatives are large plates, T-tube, or small modules

- *Reliability of the divertor target is a big feasibility issue of the power plant*

The Reference Design HEMJ* (PPCS-B, -AB, -C), for 10 MW/m²



* He-cooled modular divertor with jet cooling



9-Finger module

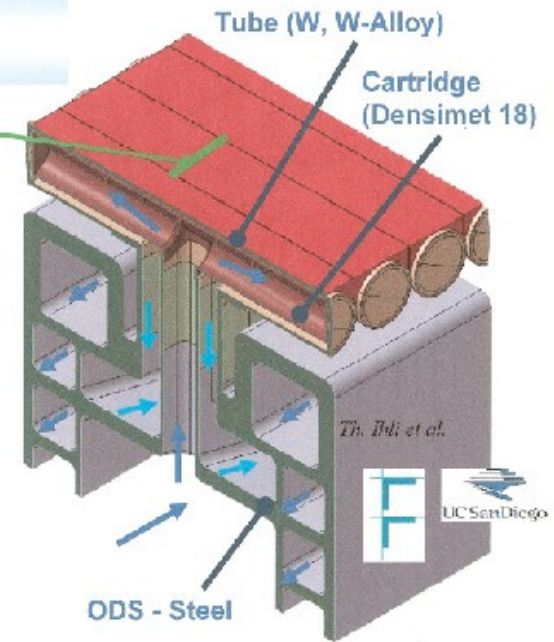
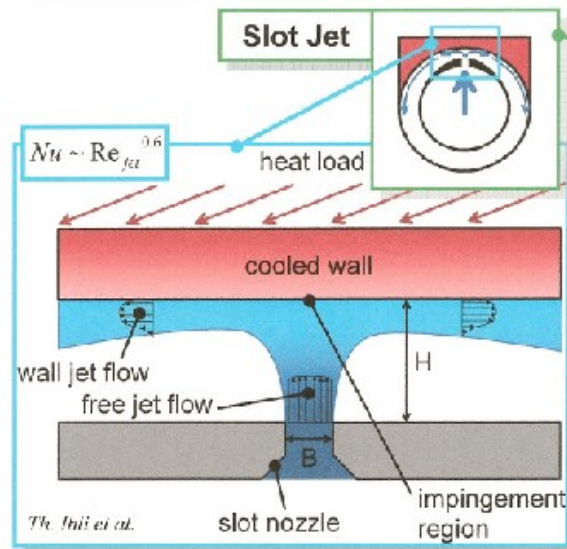
1-Finger module

Advanced ARIES T-Tube Divertor

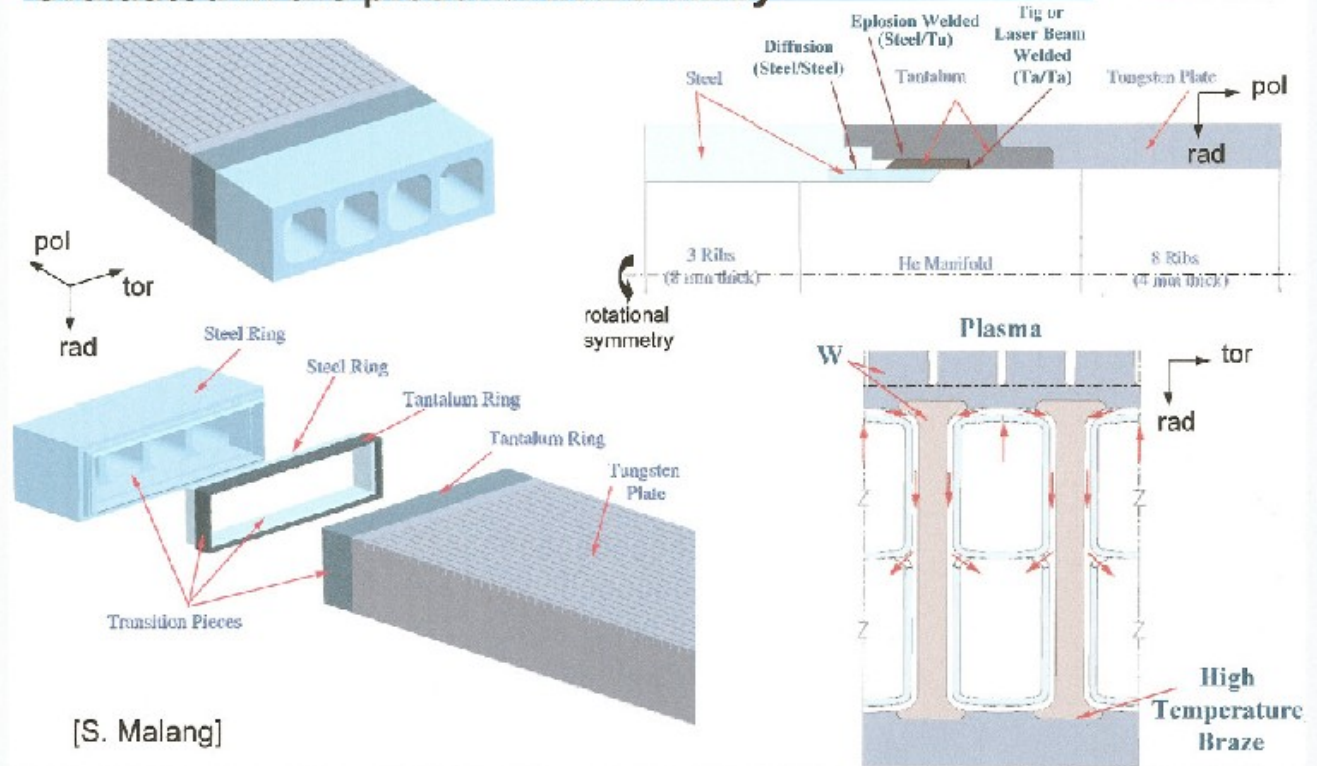


ARIES T-Tube Divertor

Jet impingement Cooling



Outlook (2): „Modified plate divertor concept“ to be evaluated in the present ARIES study



[S. Malang]

Comparison of the different design options (plate, T-tube, small modules) for helium cooled divertor targets

- a) For a heat flux of 10 MW/m^2 the temperature of divertor structure can be kept inside the anticipated temperature window for tungsten alloys (700 C to 1300 C).
- b) Stresses and deformations caused by the temperature variation in the tungsten structure are smallest for the small modules and largest for the plate design.
- c) The stresses in the plate design depend to a large degree on the kind of mechanical plate support. They assume a minimum value if the plate is allowed to bend and a maximum value if any bending is suppressed.
- d) For a $1000 \text{ MW}_{\text{el}}$ power plant the number of target elements is **500 plates** or **50,000 T-tubes** or **250,000 modules**