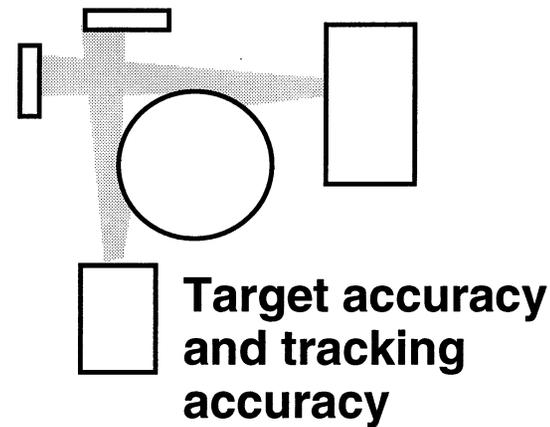
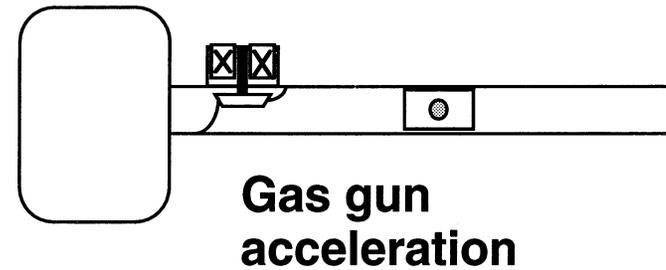
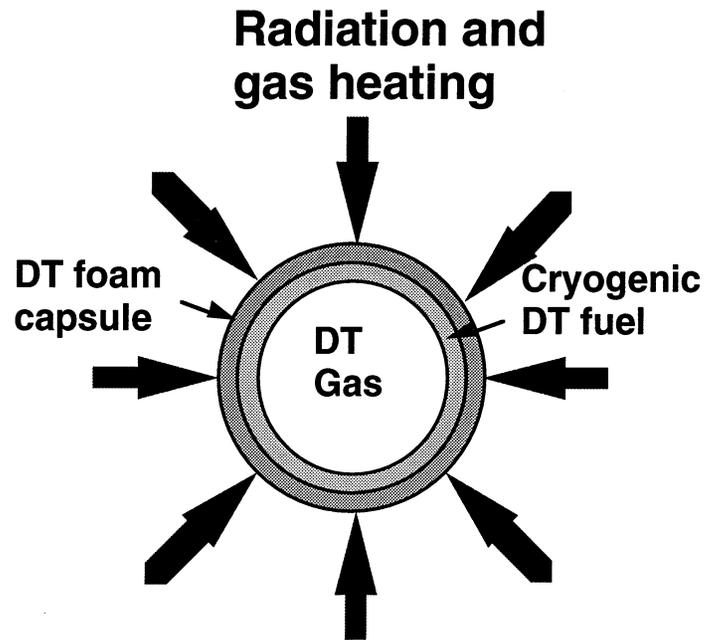


**APPENDIX I**  
**TARGET INJECTION AND TRACKING**  
**BY**  
**RON PETZOLDT**

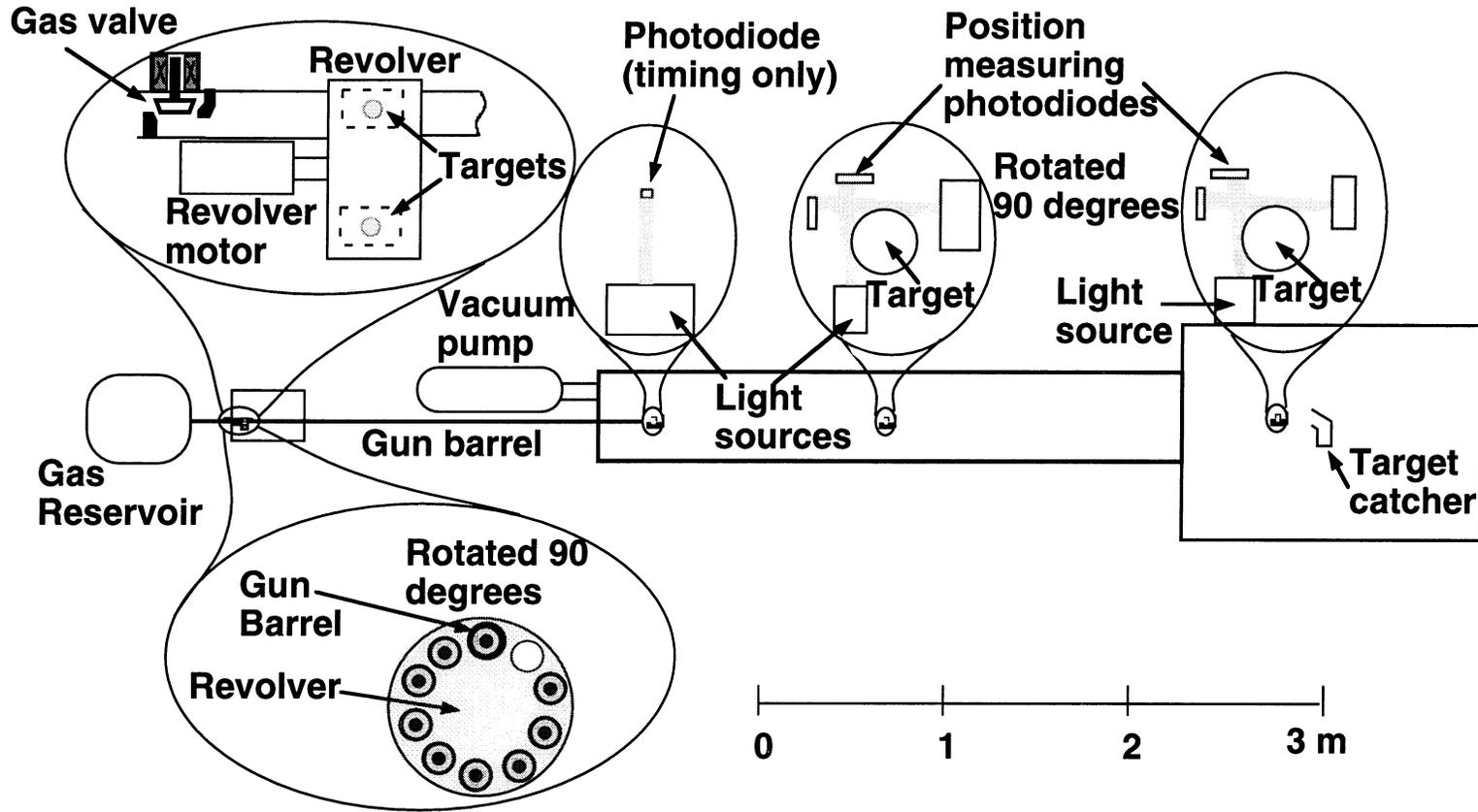
# Direct drive target injection and tracking issues for IFE



**Ron Petzoldt**  
**Direct Drive Target Workshop**  
**15 September 1999**



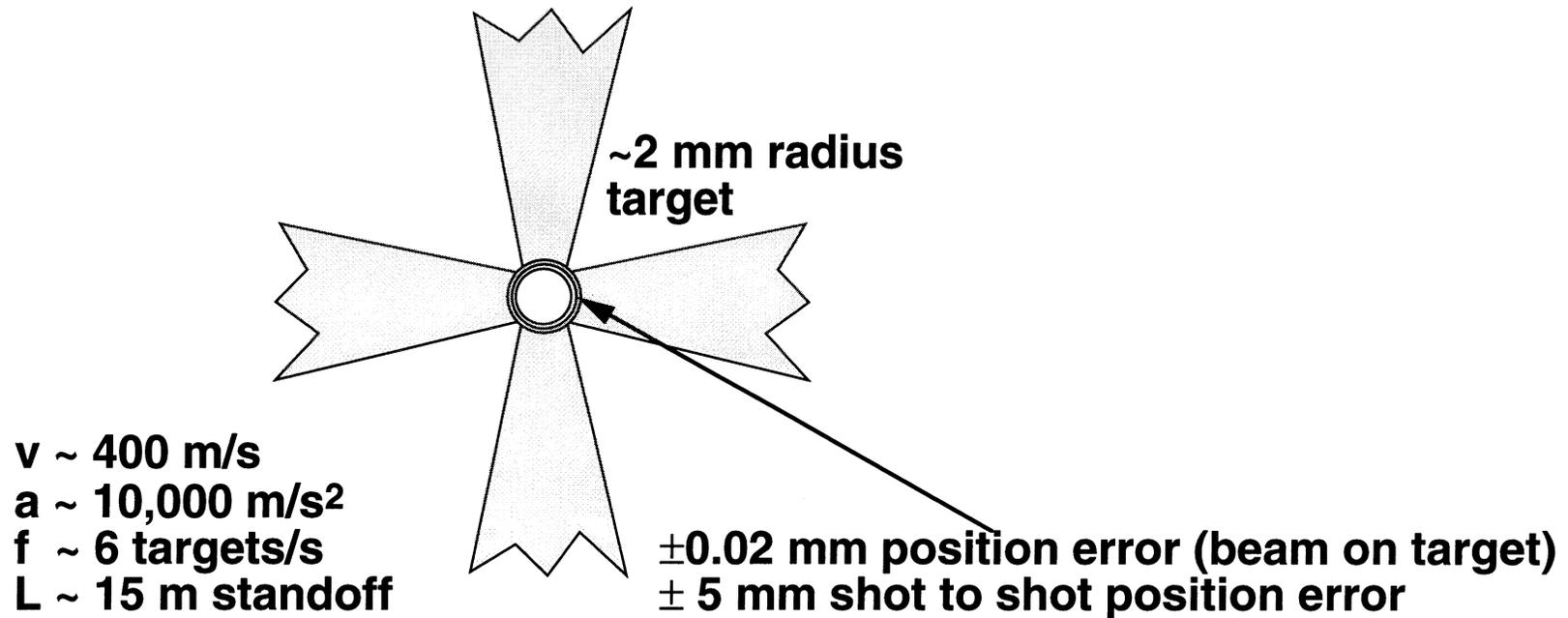
# We have demonstrated the feasibility of accurate indirect drive target injection and tracking



Ref:  
IFE Target Injection and Tracking Experiment  
*Fusion Technology* Vol. 34 Nov. 1998 pp. 831-839

# Targets must be precisely tracked into the reaction chamber at a speed of about 400 m/s

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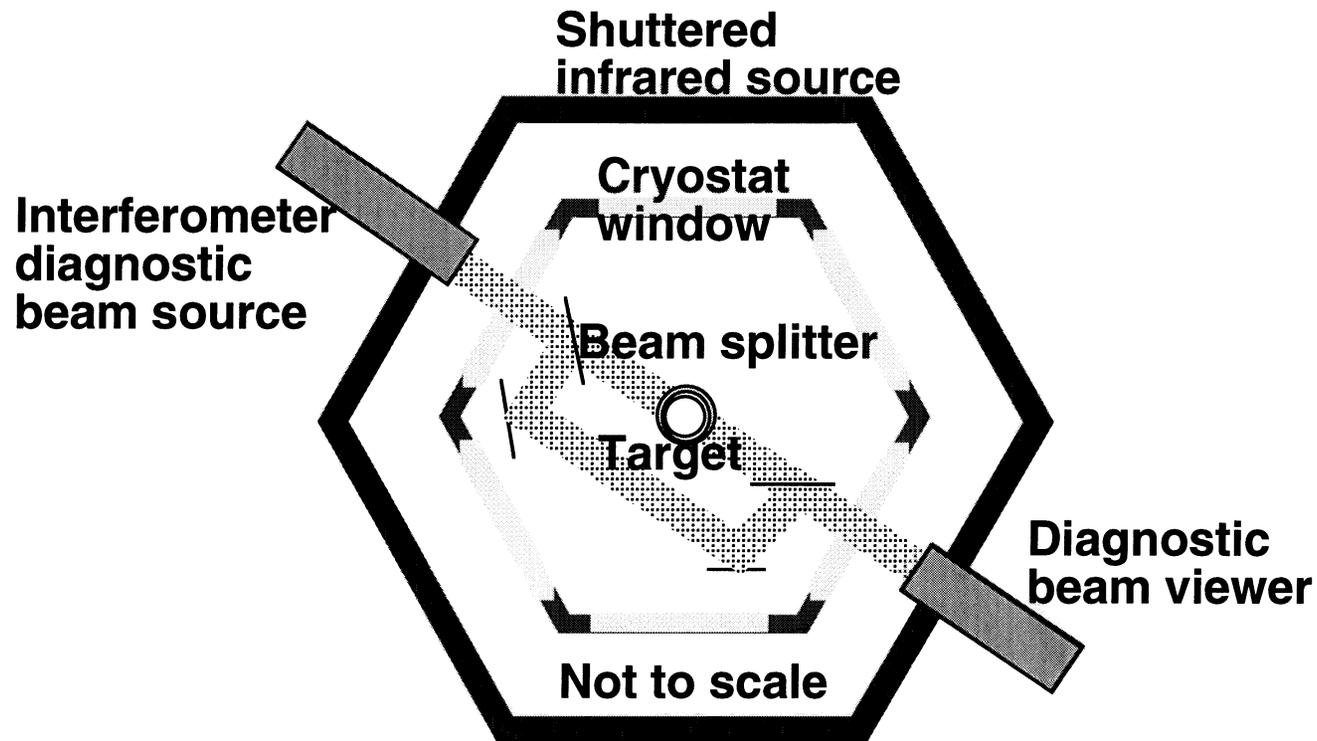


Total error of  $\pm 0.02 \text{ mm}$  { Target tracking error  $\pm 0.014 \text{ mm}$   
Beam pointing error  $\pm 0.014 \text{ mm}$

Cryogenic fuel temperature rise  $\sim 0.5 \text{ K}$  at surface

# Target response to a surface heat load should be measured

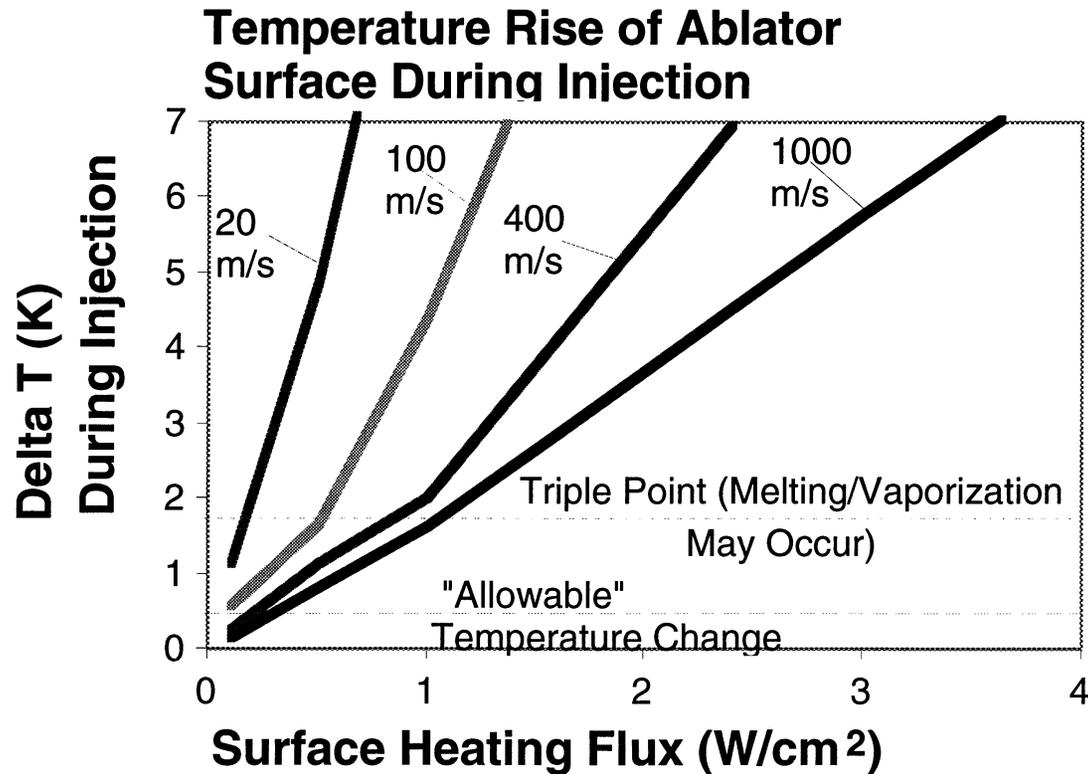
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$$\begin{aligned} \dot{q}_{rad} &= \epsilon \sigma T^4 \\ &\approx (0.01)(5.67 \times 10^{-12} \text{ W / cm}^2 \text{ K}^4)(1758 \text{ K})^4 \\ &= 0.54 \text{ W / cm}^2 \end{aligned}$$

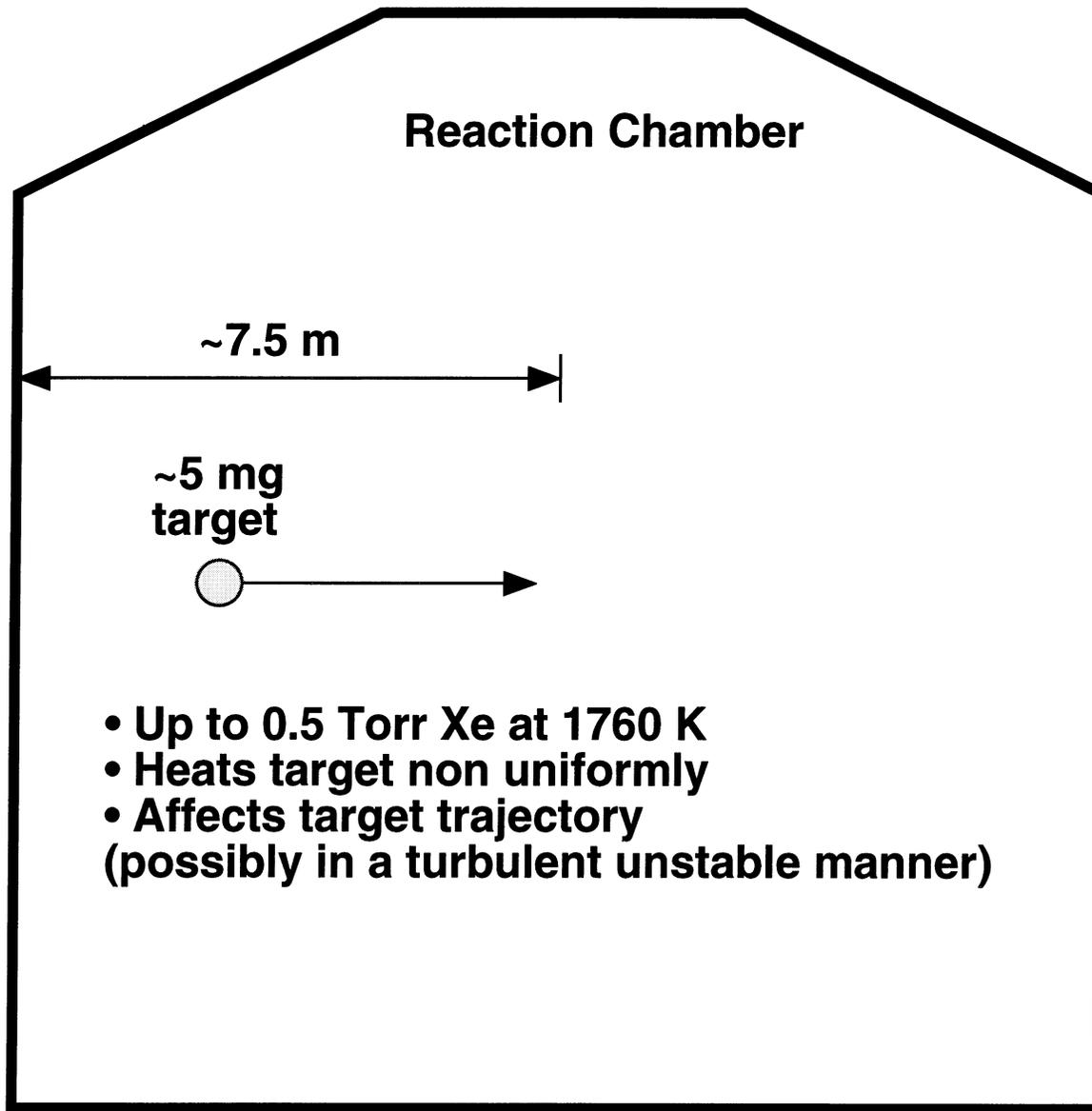
# For constant heat flux, final surface temperature decreases little as speed increases above 400 m/s

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(Assumes 7 m chamber radius with  
NRL radiation preheat target)

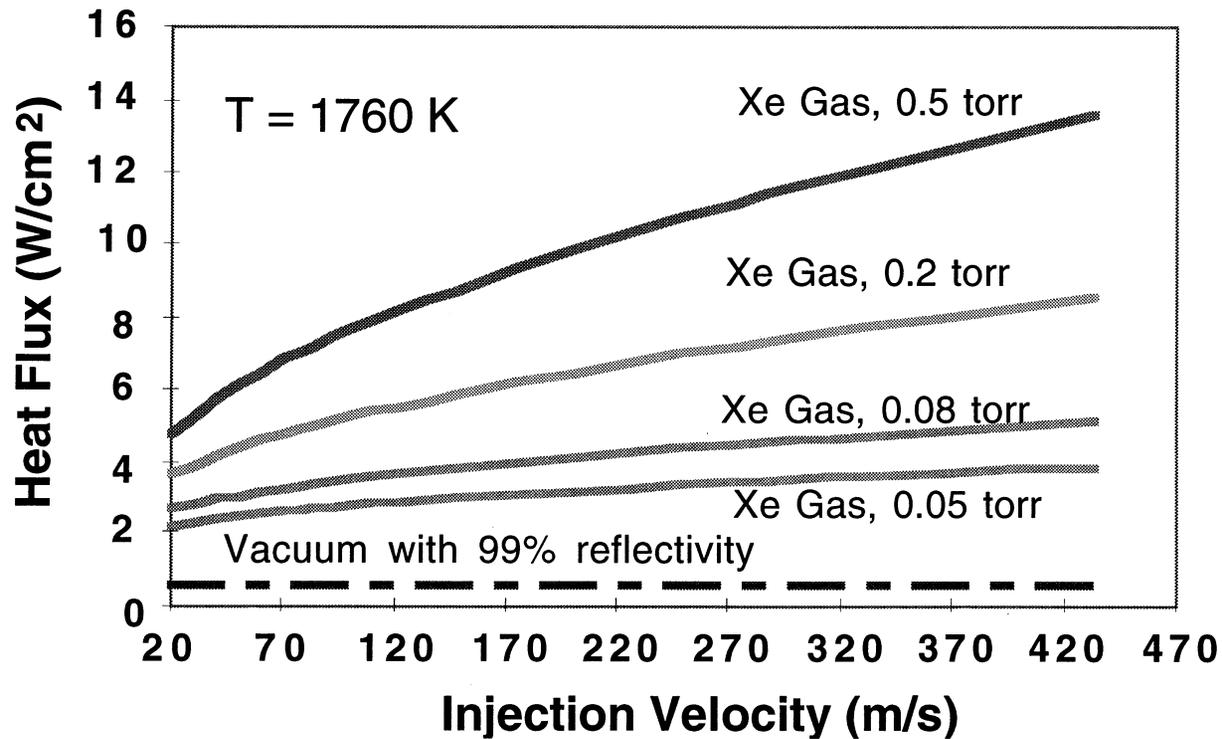
# Gas proposed to protect chamber walls will adversely affect direct drive targets



# Gas heat loads are too high, even at 0.05 Torr, and increase with injection speed

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Incident Heat Flux Using Whitaker Continuum Equation With Slip Flow Correction



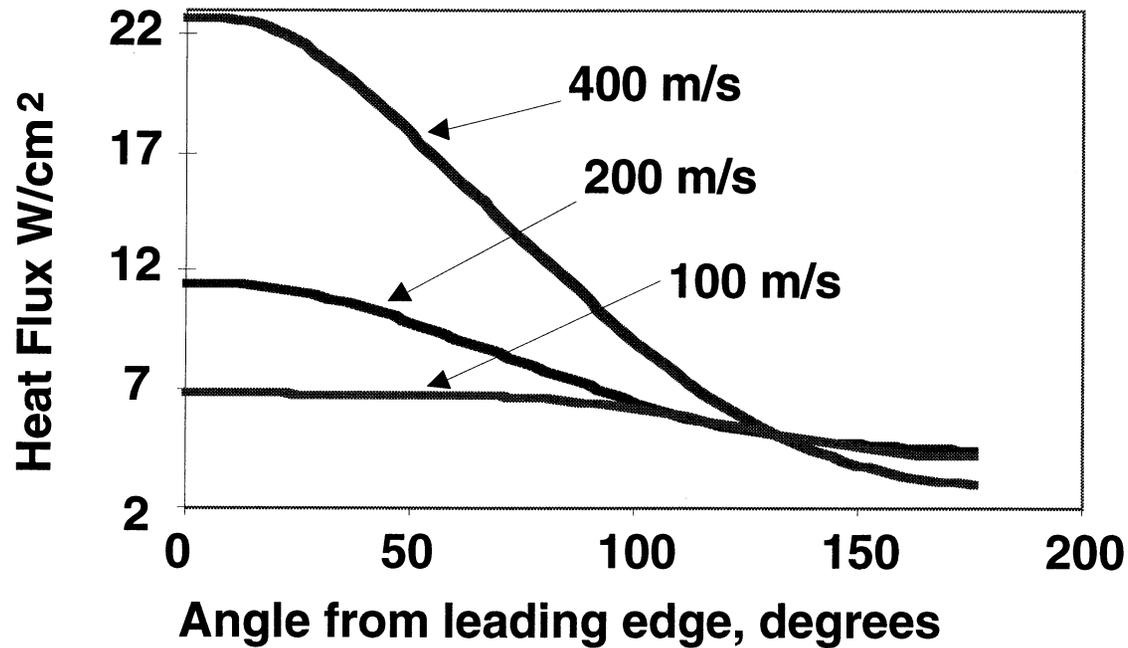
Gas heat flux

# Gas heat loads are quite asymmetric

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Heating distribution for direct-drive IFE target

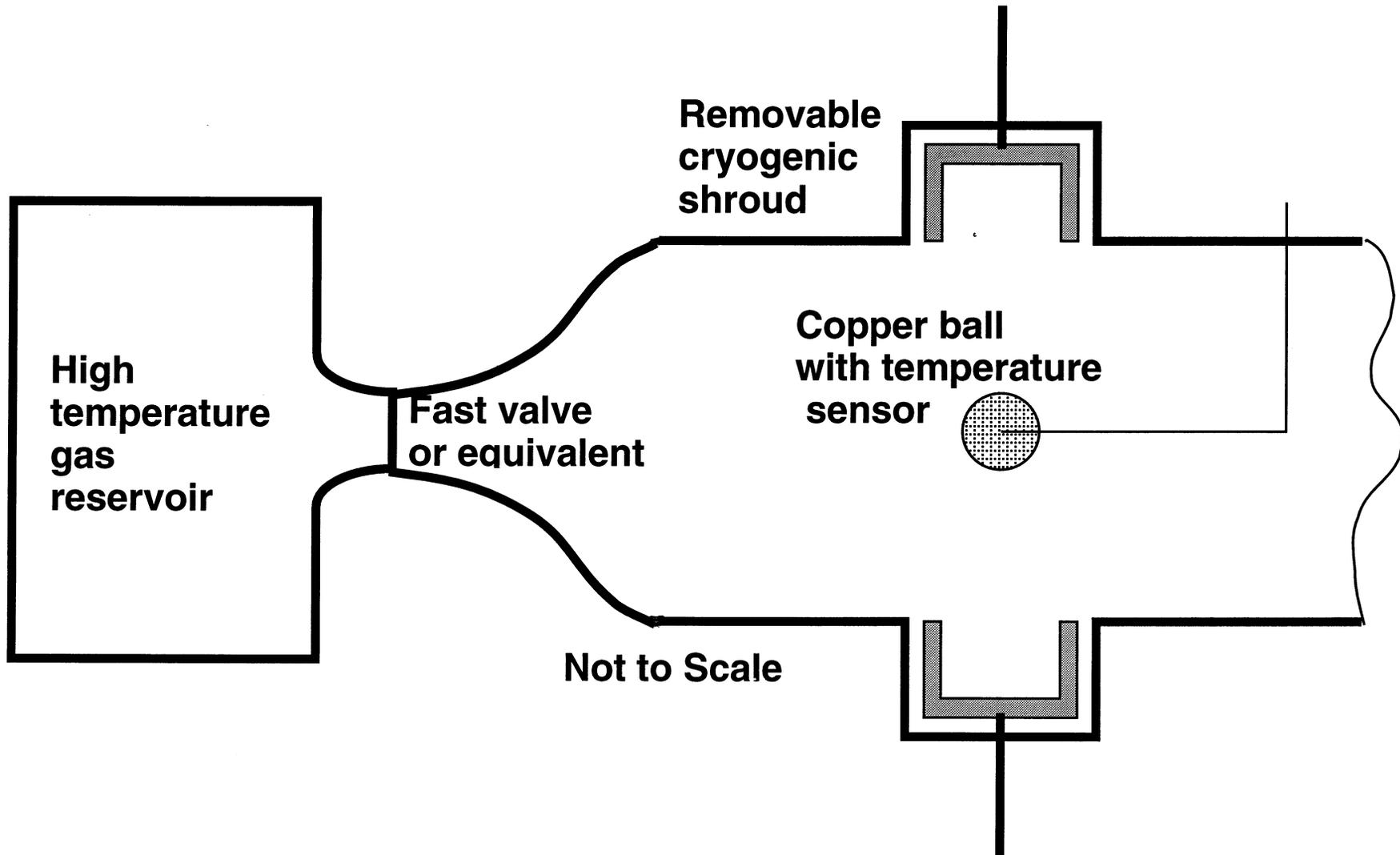
Diameter= 3.904 mm, T=1758 K, P xenon=0.5 torr



Heat flux asymmetry

# Gas heat flux calculations should be experimentally verified (if feasible)

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# Characteristic times for gas heating experiment look good

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Assume 1 cm radius spherical copper ball at 20 K  
with about  $1 \text{ W} / \text{cm}^2$  average heat flux

$$\rho = 9 \text{ g} / \text{cm}^3$$

$$k = 105 \text{ W} / \text{cm} \cdot \text{K} \text{ at } 20 \text{ K}; 68 \text{ W} / \text{cm} \cdot \text{K} \text{ at } 25 \text{ K}$$

$$c_p = 7 \text{ mJ} / \text{g} \cdot \text{K} \text{ at } 20 \text{ K}; 15 \text{ mJ} / \text{g} \cdot \text{K} \text{ at } 25 \text{ K}$$

$$\alpha = k / \rho c = 1600 \text{ cm}^2 / \text{s} \text{ at } 20 \text{ K}$$

Characteristic thermal diffusion time

$$t_d = r^2 / \alpha = \frac{(1 \text{ cm})^2}{1600 \text{ cm}^2 / \text{s}} = 0.7 \text{ ms}$$

$$m = \rho(4/3)\pi r^3 = 38 \text{ g}$$

$$E = mc\Delta T = (38 \text{ g})(7 \text{ mJ} / \text{g} \cdot \text{K})(1 \text{ K}) = 0.26 \text{ J}$$

(for 1 K temperature increase)

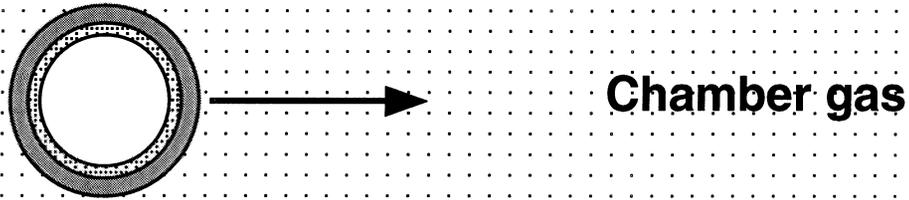
$$P = \phi 4\pi r^2 = (1 \text{ W} / \text{cm}^2)(4)(3.14)(1 \text{ cm})^2 = 12.6 \text{ W}$$

Time for 1 K temperature increase

$$t_{1\text{K}} = E / P = 0.26 \text{ J} / 12.6 \text{ W} = 21 \text{ ms}$$

# Chamber protection gas significantly affects direct drive target trajectory

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The mass of gas in the target path is

$$m = \pi r_T^2 \rho r_c = 3.14 (0.2 \text{ cm})^2 (4 \times 10^{-3} \text{ mg/cm}^3) (750 \text{ cm}) = 0.38 \text{ mg}$$

The change in target position due to gas slowing in the chamber is

$$r_c \frac{\Delta v_{ave}}{v_{ave}} = r_c \frac{v_i - v_f}{2v_{ave}} \approx r_c \frac{m_g}{2(m_g + m_t)} = 750 \text{ cm} \frac{0.38}{10} = 28 \text{ cm}$$

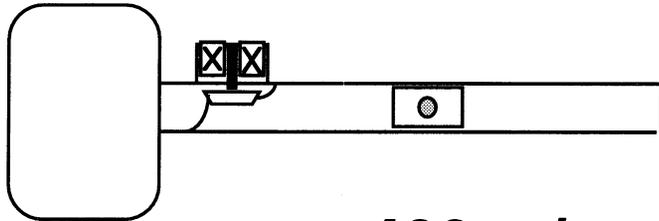
# **There are possibilities for reducing the chamber gas pressure.**

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- **Current target designs have reduced x-ray emissions**
- **Plasma debris could be redirected by magnetic fields**
- **Liquid surfaces could reduce the need for protective gases**

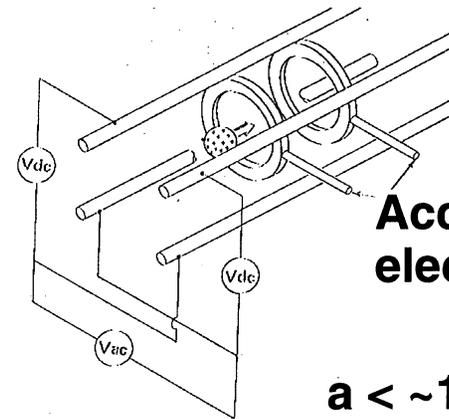
# Among several possibilities, the gas gun is our recommended experimental target injector

Gas gun



$v < \sim 400 \text{ m/s}$

Electrostatic accelerator

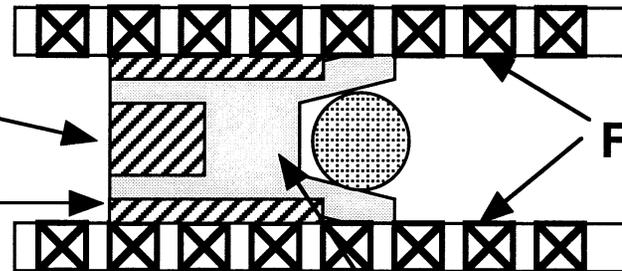


Accelerating electrodes

$a < \sim 1,000 \text{ m/s}^2$

Magnetic accelerator

Ferromagnetic cylinder or conducting ring



Field coils

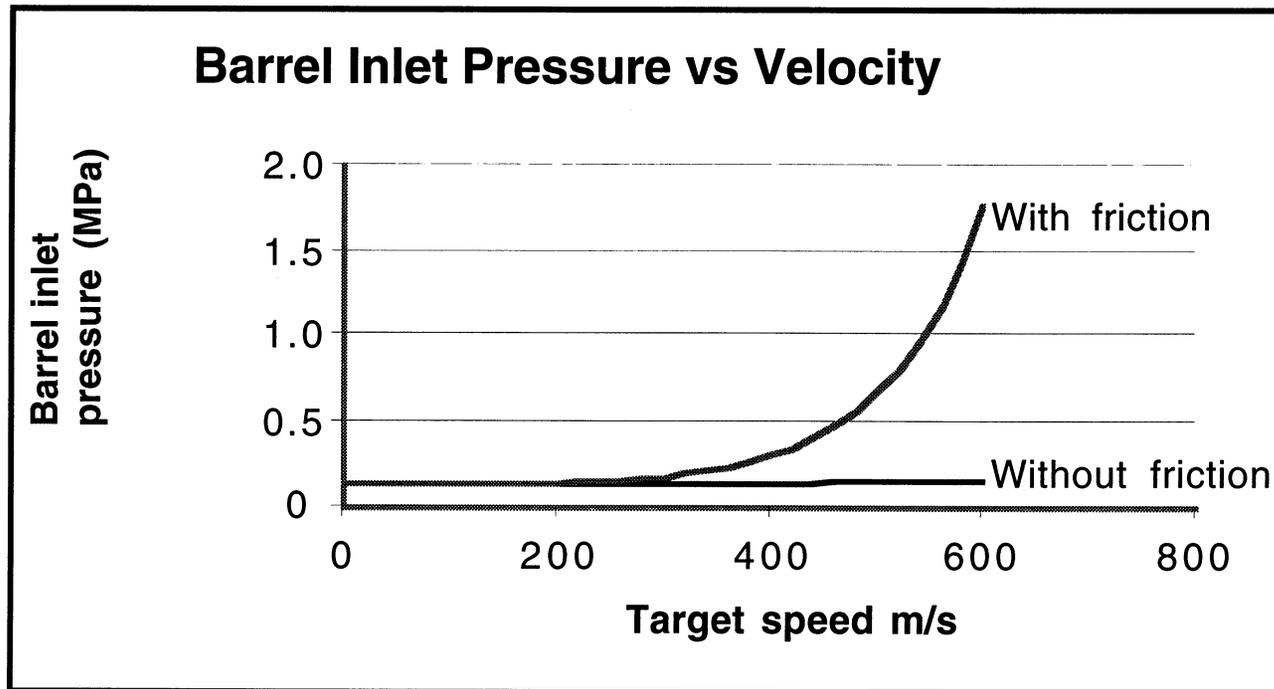
Sabot

# Gas friction increases faster than $u^4$ This limits max feasible speed

$$F_{fg} = \frac{m_g c_f u^2}{r_p} \quad c_f \approx 0.049 N_{Re}^{-1/5} \approx 0.005$$

$$p_i = p_a + p_f = p_0 + \frac{m_g \alpha}{A} + \frac{F_f}{A} = p_0 + \frac{m_g}{\pi r_p^2} \left( \alpha + \frac{c_f u^2}{r_p} \right)$$

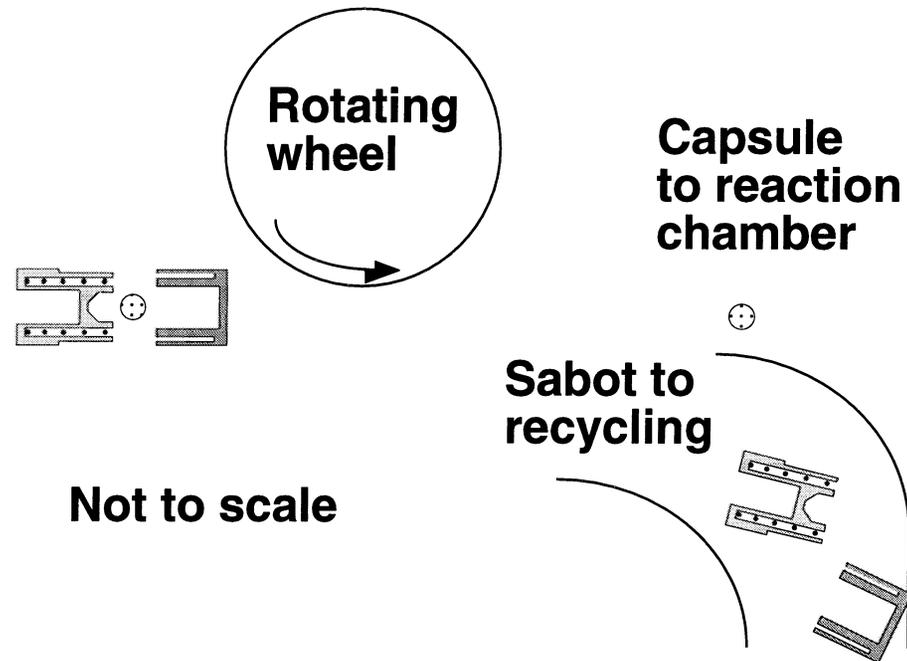
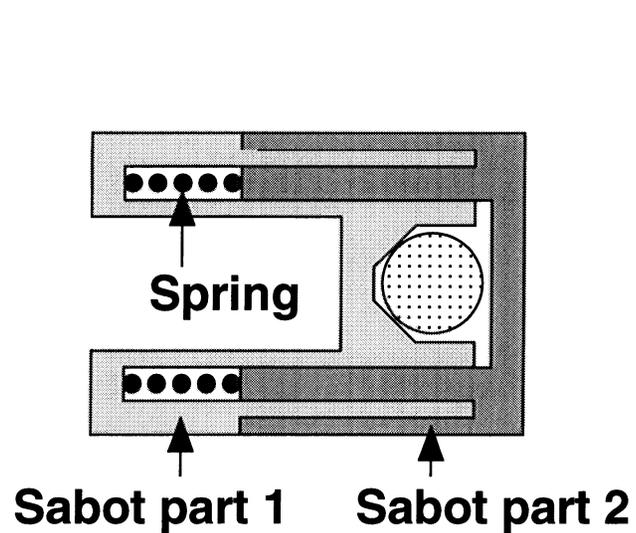
$$m_g = \int \rho dV = \pi r_p^2 \int_0^{x_p} \rho dx \quad \rho = \rho_0 \left( \frac{p}{p_0} \right)^{\frac{1}{\gamma}}$$



**1 g, 1 cm diameter target with 10,000 m/s<sup>2</sup> acceleration  
by room temperature hydrogen**

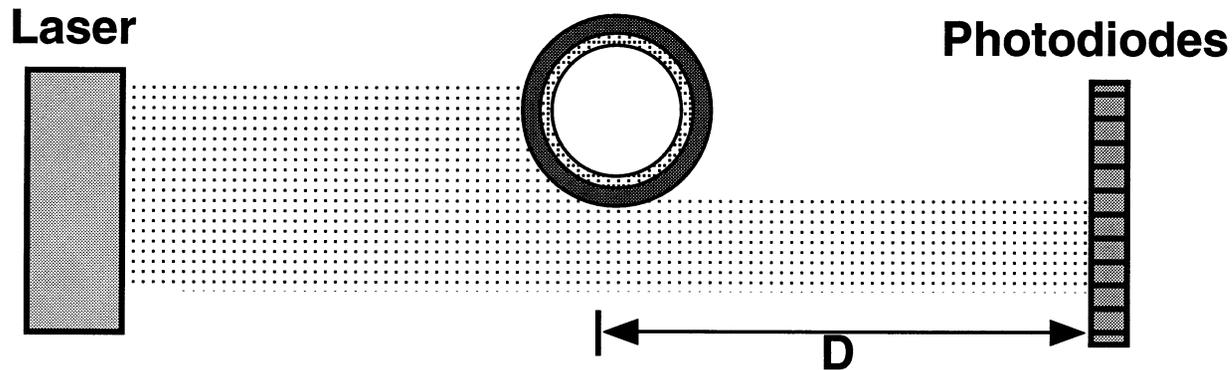
# A spring loaded sabot could protect a capsule from gas heating during and immediately after acceleration

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# Photodiode detector technology might achieve the required direct drive tracking accuracy

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Measuring voltage output from several small photodiodes will increase position measurement sensitivity.

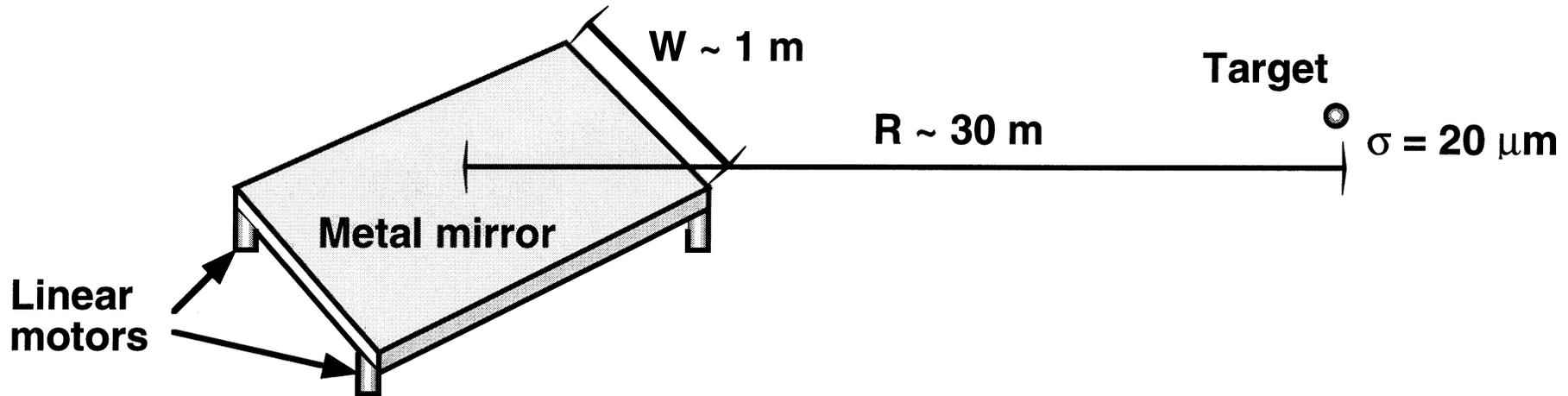
Sensor position control and/or feedback will be critical.

Required accuracy has yet to be demonstrated.

Diffraction will be important especially for large D  
Shadow washout  $d \sim \sqrt{D\lambda}$  ? I need to review this more.

## Linear motors with grazing incidence mirrors should provide adequate beam position control.

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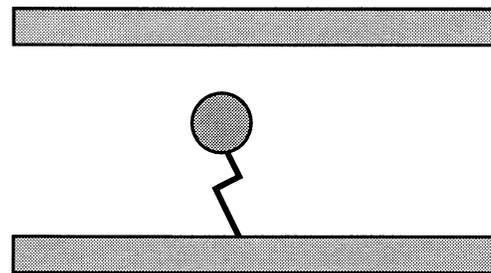
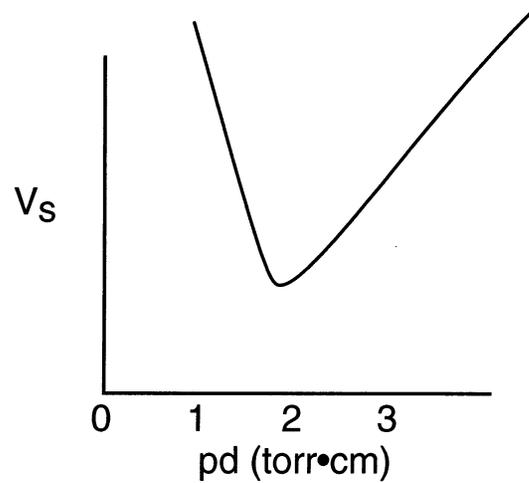
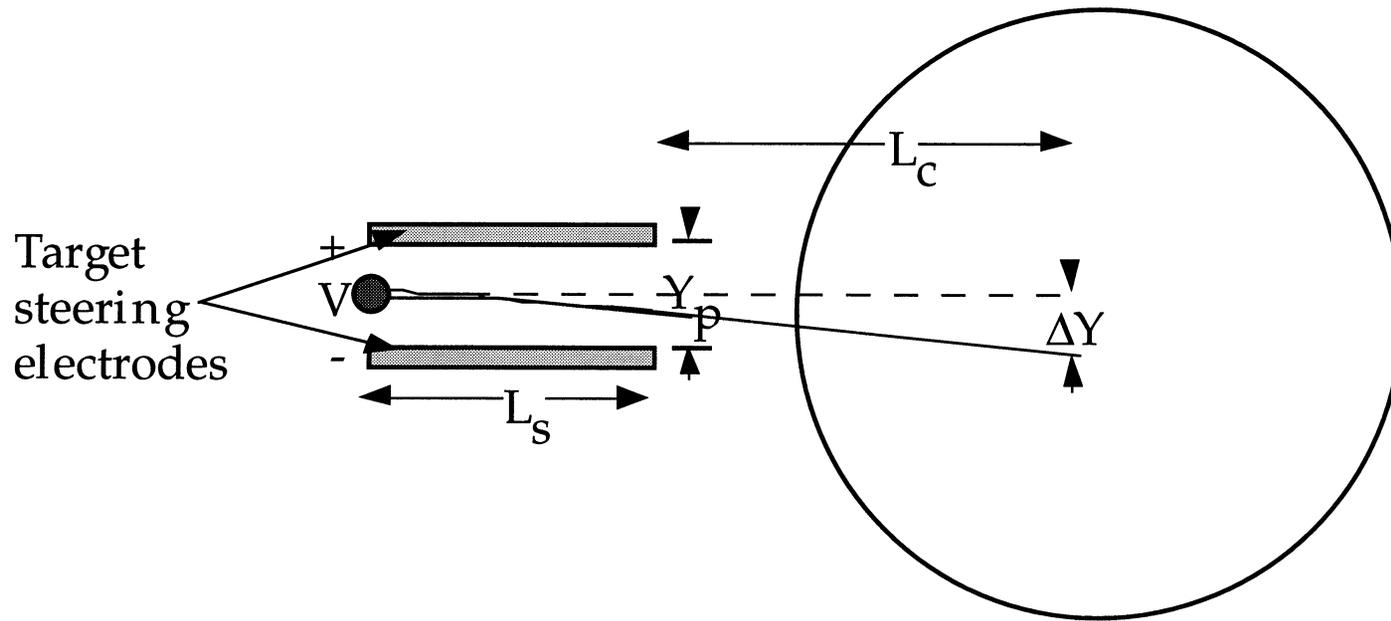
$$\text{Required motor position accuracy} \approx \frac{W \sigma}{2 R} \approx \frac{1 \text{ m } 20 \text{ } \mu\text{m}}{2 \cdot 30 \text{ m}} = 0.3 \text{ } \mu\text{m}$$

is well within state of the art motor accuracy of  $0.02 \text{ } \mu\text{m}$ .  
Required forces and response time are also within motor capabilities.

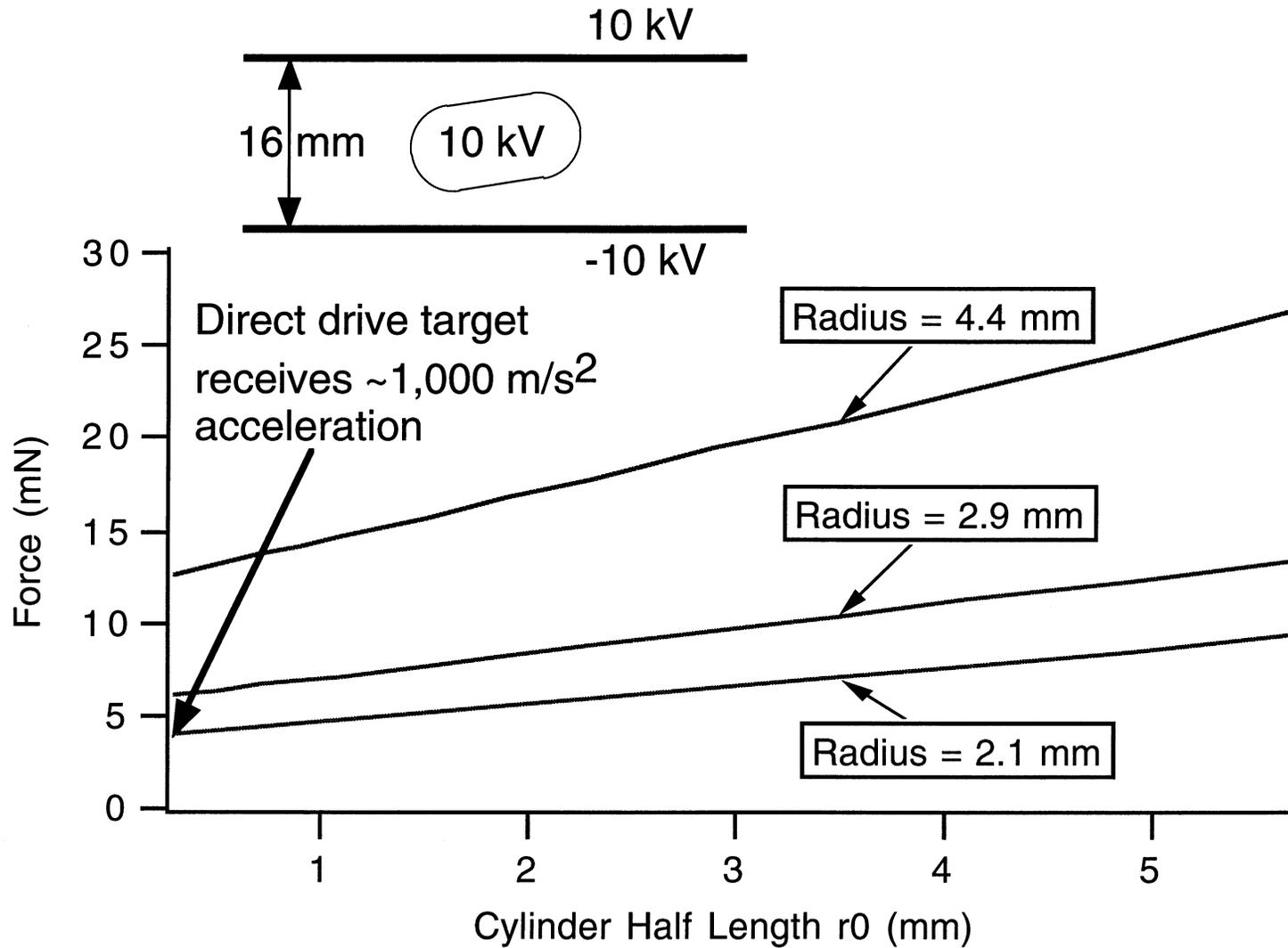
Remaining concerns for beam steering include position calibration and feed back.

Repositioning smaller optics early in beam might be preferred.

# Electrostatic target steering is possible but electrical breakdown can occur especially with gases present



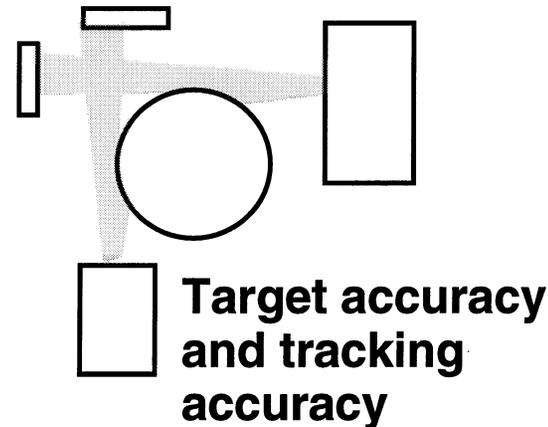
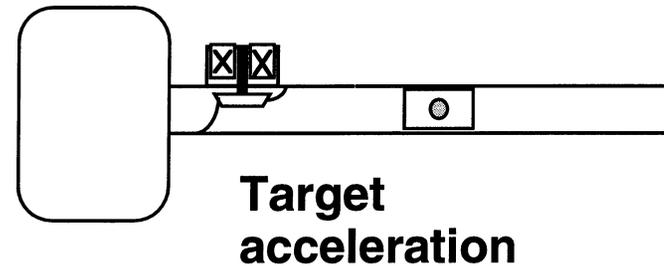
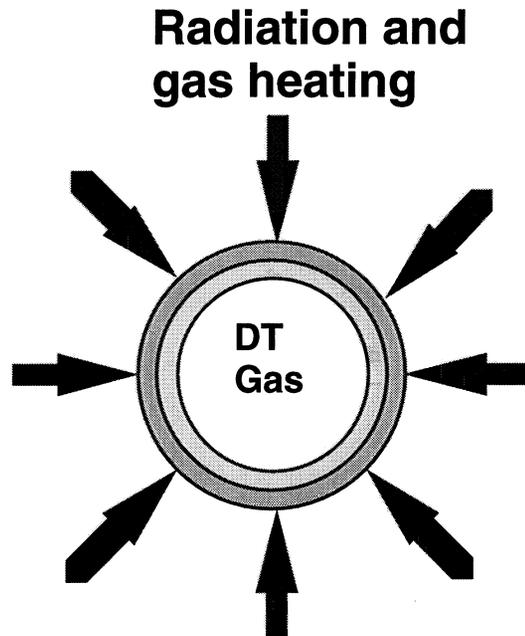
# Target steering force due to 10 kV potentials was calculated



## Conclusions:

Three significant issues facing direct drive IFE are target temperature limits and control, target injection and tracking accuracy, and allowable target acceleration. Resolving these issues will require significant analytical and experimental work.

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# CONCEPTUAL LAYOUT OF DT STRENGTH MEASUREMENT EQUIPMENT

